

13.4. IDENTIFY: The period is the time for one cycle and the amplitude is the maximum displacement from equilibrium. Both these values can be read from the graph.

SET UP: The maximum x is 10.0 cm. The time for one cycle is 16.0 s.

EXECUTE: (a) $T = 16.0$ s so $f = \frac{1}{T} = 0.0625$ Hz.

(b) $A = 10.0$ cm.

(c) $T = 16.0$ s

(d) $\omega = 2\pi f = 0.393$ rad/s

EVALUATE: After one cycle the motion repeats.

13.11. IDENTIFY: Use Eq.(13.19) to calculate A . The initial position and velocity of the block determine ϕ . $x(t)$ is given by Eq.(13.13).

SET UP: $\cos\theta$ is zero when $\theta = \pm\pi/2$ and $\sin(\pi/2) = 1$.

EXECUTE: (a) From Eq. (13.19), $A = \left| \frac{v_0}{\omega} \right| = \left| \frac{v_0}{\sqrt{k/m}} \right| = 0.98$ m.

(b) Since $x(0) = 0$, Eq.(13.14) requires $\phi = \pm\frac{\pi}{2}$. Since the block is initially moving to the left, $v_{0x} < 0$ and Eq.(13.7)

requires that $\sin\phi > 0$, so $\phi = +\frac{\pi}{2}$.

(c) $\cos(\omega t + (\pi/2)) = -\sin\omega t$, so $x = (-0.98 \text{ m}) \sin((12.2 \text{ rad/s})t)$.

EVALUATE: The $x(t)$ result in part (c) does give $x = 0$ at $t = 0$ and $x < 0$ for t slightly greater than zero.

13.15. IDENTIFY: Apply $T = 2\pi\sqrt{\frac{m}{k}}$. Use the information about the empty chair to calculate k .

SET UP: When $m = 42.5$ kg, $T = 1.30$ s.

EXECUTE: Empty chair: $T = 2\pi\sqrt{\frac{m}{k}}$ gives $k = \frac{4\pi^2 m}{T^2} = \frac{4\pi^2(42.5 \text{ kg})}{(1.30 \text{ s})^2} = 993 \text{ N/m}$

With person in chair: $T = 2\pi\sqrt{\frac{m}{k}}$ gives $m = \frac{T^2 k}{4\pi^2} = \frac{(2.54 \text{ s})^2(993 \text{ N/m})}{4\pi^2} = 162 \text{ kg}$ and

$m_{\text{person}} = 162 \text{ kg} - 42.5 \text{ kg} = 120 \text{ kg}$.

EVALUATE: For the same spring, when the mass increases, the period increases.

13.19. IDENTIFY: Compare the specific $x(t)$ given in the problem to the general form of Eq.(13.13).

SET UP: $A = 7.40$ cm, $\omega = 4.16$ rad/s, and $\phi = -2.42$ rad.

EXECUTE: (a) $T = \frac{2\pi}{\omega} = \frac{2\pi}{4.16 \text{ rad/s}} = 1.51$ s.

(b) $\omega = \sqrt{\frac{k}{m}}$ so $k = m\omega^2 = (1.50 \text{ kg})(4.16 \text{ rad/s})^2 = 26.0 \text{ N/m}$

(c) $v_{\text{max}} = \omega A = (4.16 \text{ rad/s})(7.40 \text{ cm}) = 30.8 \text{ cm/s}$

(d) $F_x = -kx$ so $F = kA = (26.0 \text{ N/m})(0.0740 \text{ m}) = 1.92 \text{ N}$.

(e) $x(t)$ evaluated at $t = 1.00$ s gives $x = -0.0125$ m. $v_x = -\omega A \sin(\omega t + \phi) = 30.4 \text{ cm/s}$.

$a_x = -kx/m = -\omega^2 x = +0.216 \text{ m/s}^2$.

EVALUATE: The maximum speed occurs when $x = 0$ and the maximum force is when $x = \pm A$.

13.56. IDENTIFY: If the system is critically damped or overdamped it doesn't oscillate. With no damping, $\omega = \sqrt{m/k}$.

With underdamping, the angular frequency has the smaller value $\omega' = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}}$.

SET UP: $m = 2.20 \text{ kg}$, $k = 250.0 \text{ N/m}$. $T' = \frac{2\pi}{\omega'}$ and $\omega' = \frac{2\pi}{T'} = \frac{2\pi}{0.615 \text{ s}} = 10.22 \text{ rad/s}$.

EXECUTE: (a) $\omega = \sqrt{\frac{k}{m}} = \sqrt{\frac{250.0 \text{ N/m}}{2.20 \text{ kg}}} = 10.66 \text{ rad/s}$. $\omega' < \omega$ so the system is damped. $\omega' = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}}$ gives

$$b = 2m\sqrt{\frac{k}{m} - \omega'^2} = 2(2.20 \text{ kg})\sqrt{\frac{250.0 \text{ N/m}}{2.20 \text{ kg}} - (10.22 \text{ rad/s})^2} = 13.3 \text{ kg/s}.$$

(b) Since the motion has a period the system oscillates and is underdamped.

EVALUATE: The critical value of the damping constant is $b = 2\sqrt{km} = 2\sqrt{(250.0 \text{ N/m})(2.20 \text{ kg})} = 46.9 \text{ kg/s}$. In this problem b is much less than its critical value.

13.59. IDENTIFY: $x(t)$ is given by Eq.(13.42). $v_x = dx/dt$ and $a_x = dv_x/dt$.

SET UP: $d(\cos \omega't)/dt = -\omega' \sin \omega't$. $d(\sin \omega't)/dt = \omega' \cos \omega't$. $d(e^{-\alpha t})/dt = -\alpha e^{-\alpha t}$.

EXECUTE: (a) With $\phi = 0$, $x(0) = A$.

(b) $v_x = \frac{dx}{dt} = Ae^{-(b/2m)t} \left[-\frac{b}{2m} \cos \omega't - \omega' \sin \omega't \right]$, and at $t = 0$, $v_x = -Ab/2m$; the graph of x versus t near $t = 0$

slopes down.

(c) $a_x = \frac{dv_x}{dt} = Ae^{-(b/2m)t} \left[\left(\frac{b^2}{4m^2} - \omega'^2 \right) \cos \omega't + \frac{\omega'b}{2m} \sin \omega't \right]$, and at $t = 0$, $a_x = A \left(\frac{b^2}{4m^2} - \omega'^2 \right) = A \left(\frac{b^2}{2m^2} - \frac{k}{m} \right)$.

(Note that this is $(-bv_0 - kx_0)/m$.) This will be negative if $b < \sqrt{2km}$, zero if $b = \sqrt{2km}$ and positive if $b > \sqrt{2km}$. The graph in the three cases will be curved down, not curved, or curved up, respectively.

EVALUATE: $a_x(0) = 0$ corresponds to the situation of critical damping.