

# WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

## DESIGN OF MACHINE ELEMENTS ME-3320, B'2024

Lecture 10-11

November 2024

*Optional*



# Static failure theories

Accepted failure theories that apply to **ductile** materials:

- *Total strain energy theory*
- ● *Distortion energy theory*
- *Pure shear-stress theory*
- ● *Maximum shear-stress theory*
- *Maximum normal stress theory (limited application)*

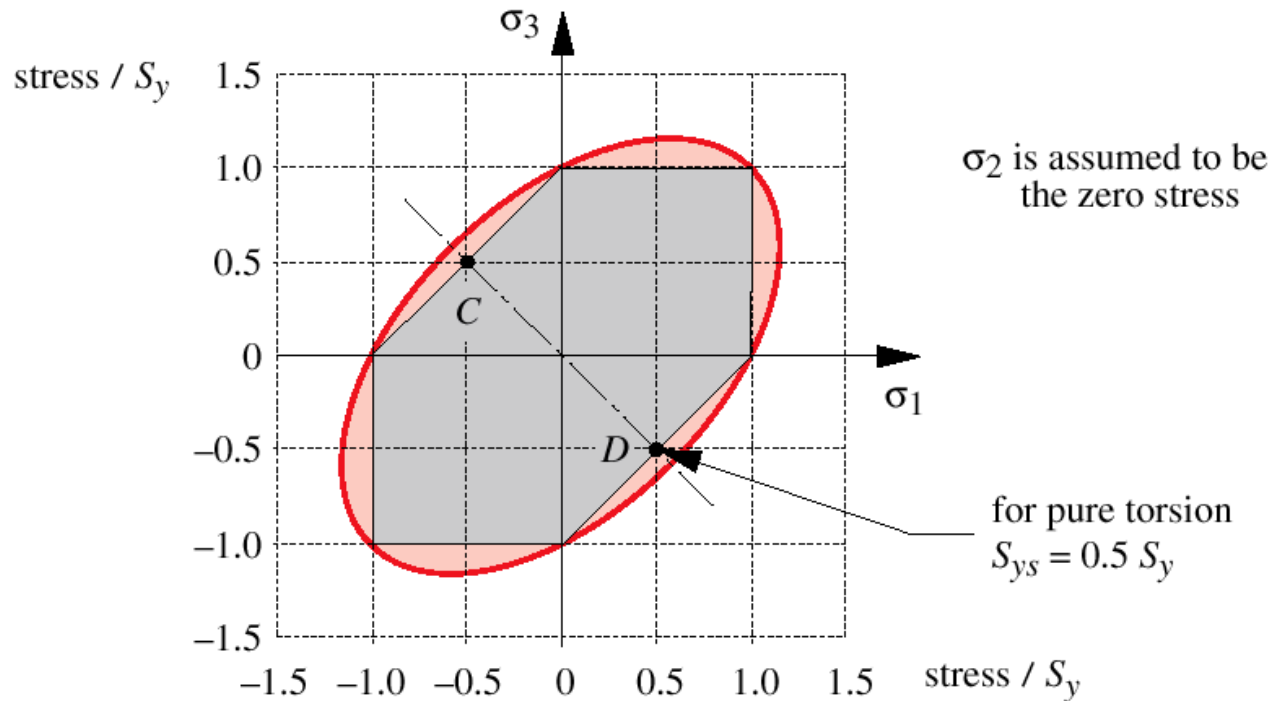
Accepted failure theories that apply to **brittle** materials:

- ● *Maximum normal stress theory (even material)*
- ● *Maximum normal stress theory (uneven material)*
- ● *Coulomb-Mohr theory*
- ● *Modified Mohr theory*



# Static failure theories

## Ductile materials



**FIGURE 5-5**

The 2-D Shear-Stress Theory Hexagon Inscribed Within the Distortion-Energy Ellipse



# Static failure theories: experimental verifications

## Ductile & brittle materials

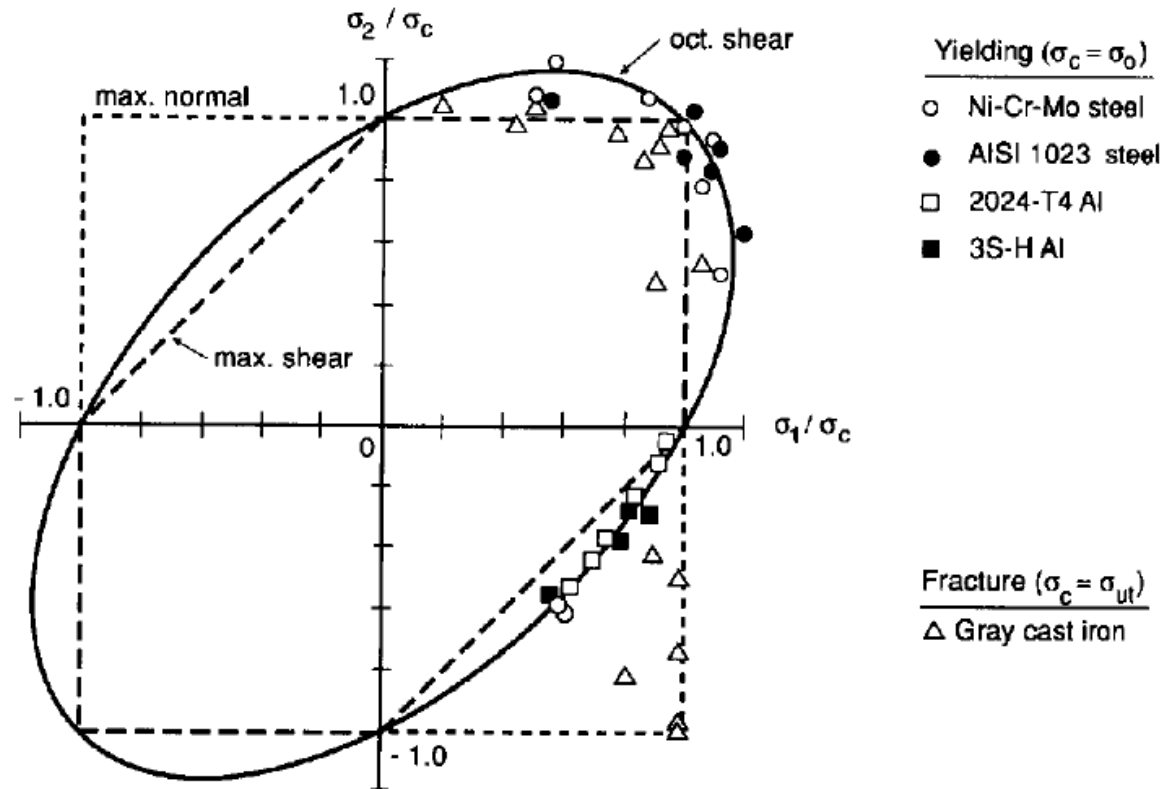


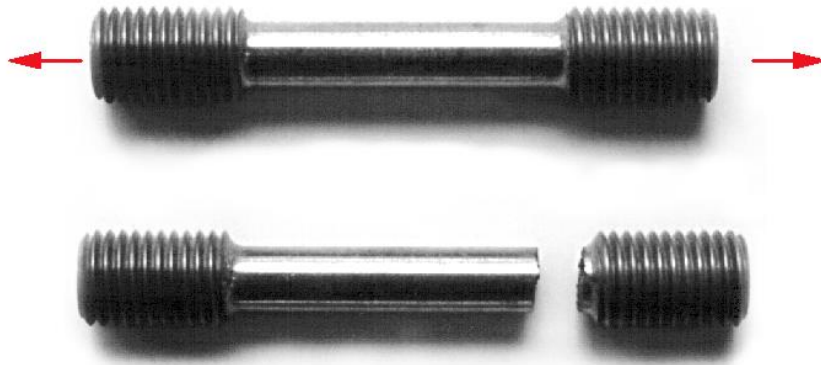
FIGURE 5-8

Experimental Data from Tensile Tests Superposed on Three Failure Theories (Reproduced from Fig. 7.11, p. 252, in *Mechanical Behavior of Materials* by N. E. Dowling, Prentice-Hall, Englewood Cliffs, NJ, 1993)



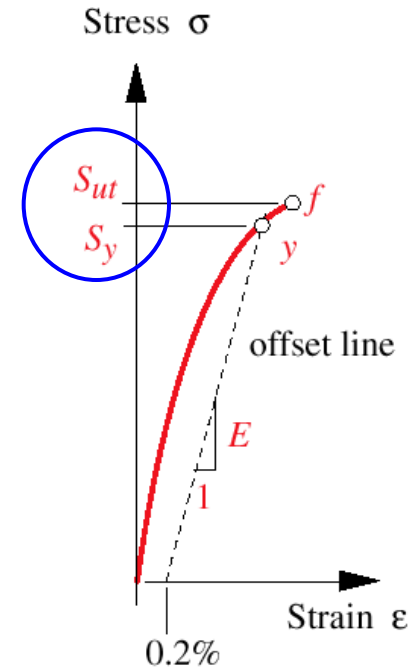
# Static failure theories

## Brittle materials



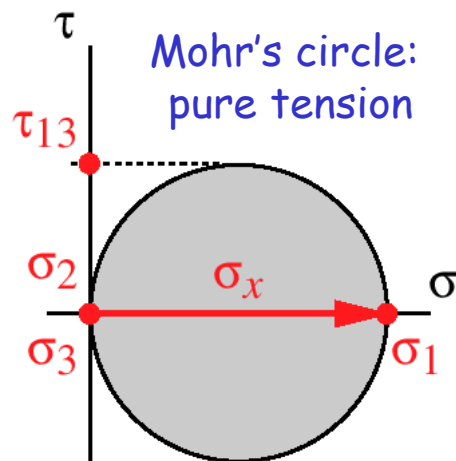
**FIGURE 2-5**

A Tensile Test Specimen of Brittle Cast Iron After Fracture



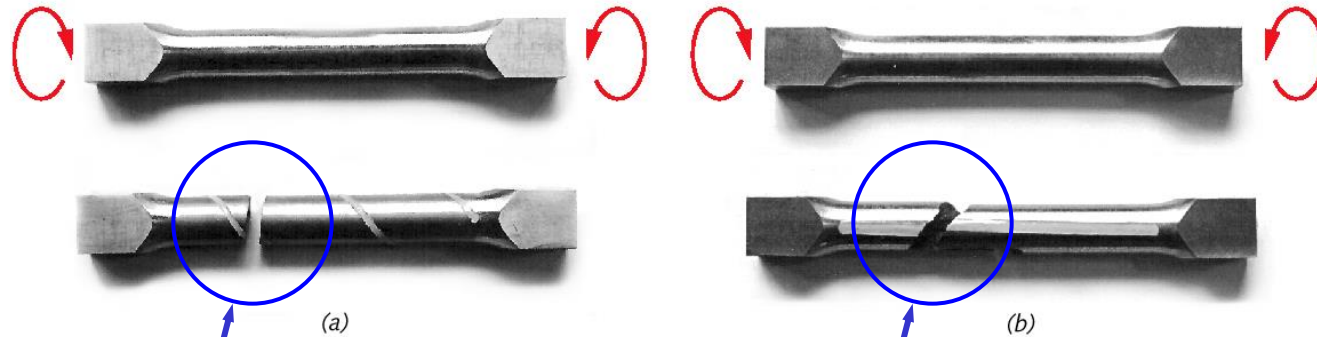
**FIGURE 2-4**

Stress-Strain Curve of a Brittle Material



# Static failure theories

## Brittle materials

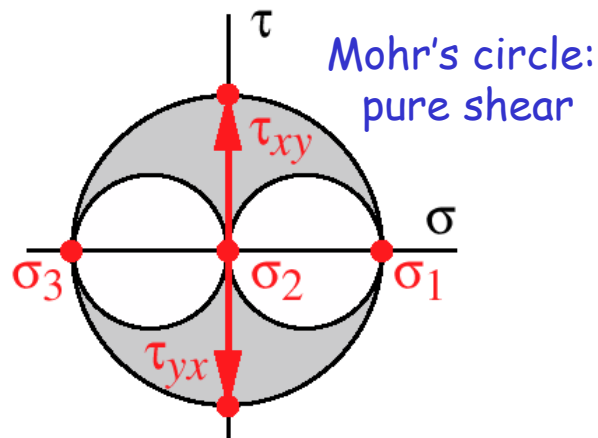


**FIGURE 2-8**

Torsion Test Specimens After Failure (a) Ductile Steel (b) Brittle Cast Iron

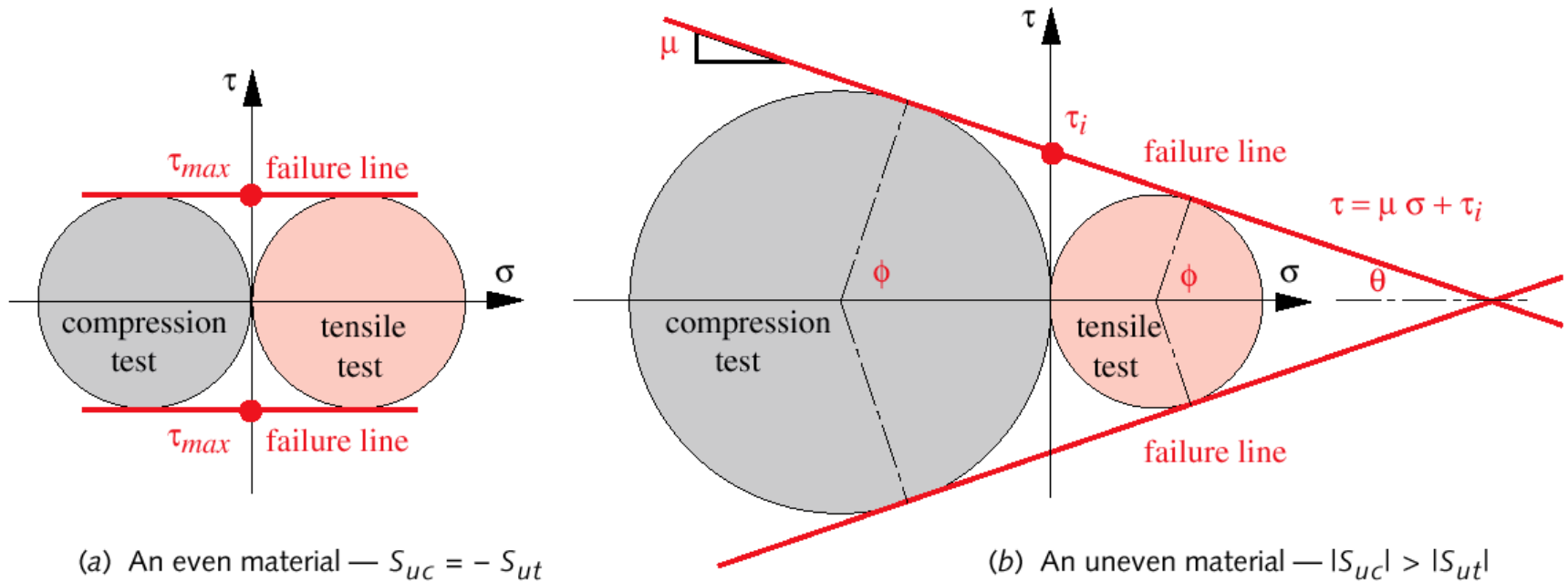
Pure shear condition

Pure shear condition



# Static failure theories

## Brittle materials: even and uneven materials



**FIGURE 5-10**

Mohr's Circles for Both Compression and Tensile Tests Showing the Failure Envelopes for (a) *Even* and (b) *Uneven* Materials



# Static failure theories

Brittle materials: *Coulomb-Mohr, modified-Mohr, and normal stress theories*

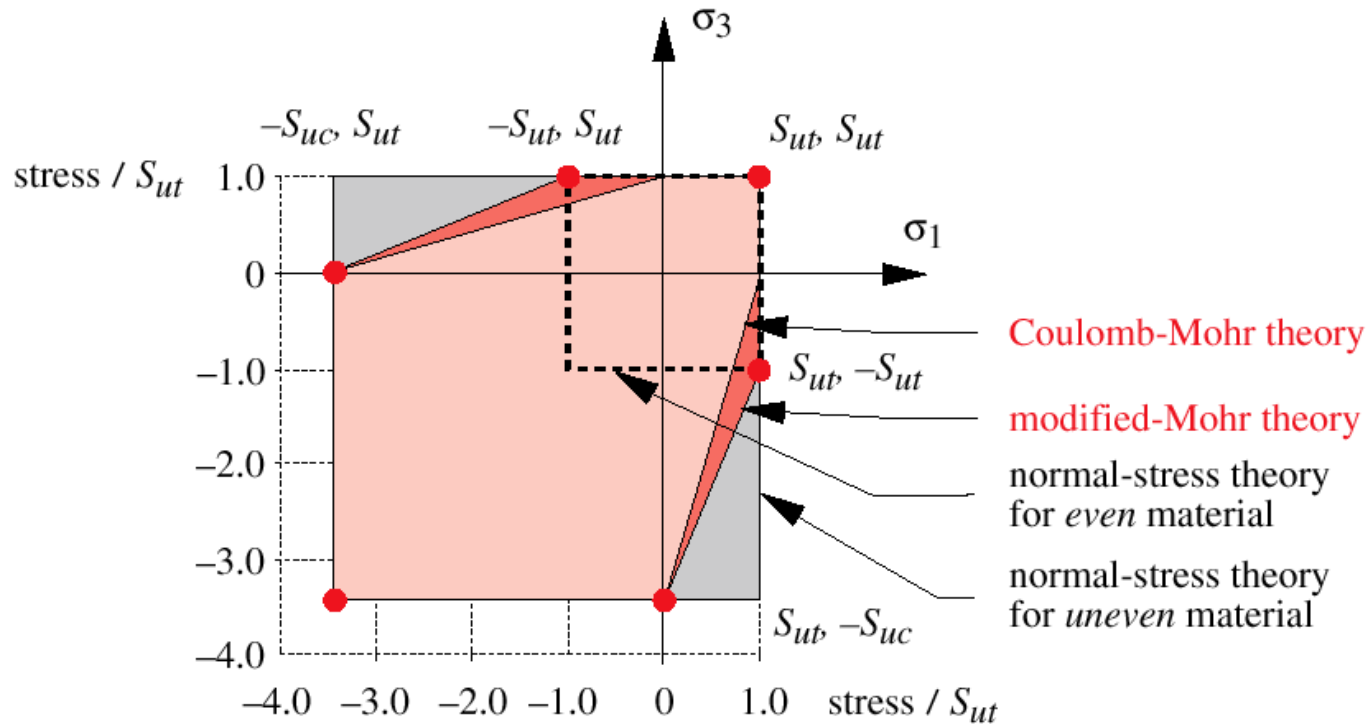


FIGURE 5-11

Coulomb-Mohr, Modified-Mohr, and Maximum Normal-Stress Theories for Uneven Brittle Materials





# Static failure theories: brittle materials

## *Coulomb-Mohr, modified-Mohr, and normal stress theories* Experimental observations

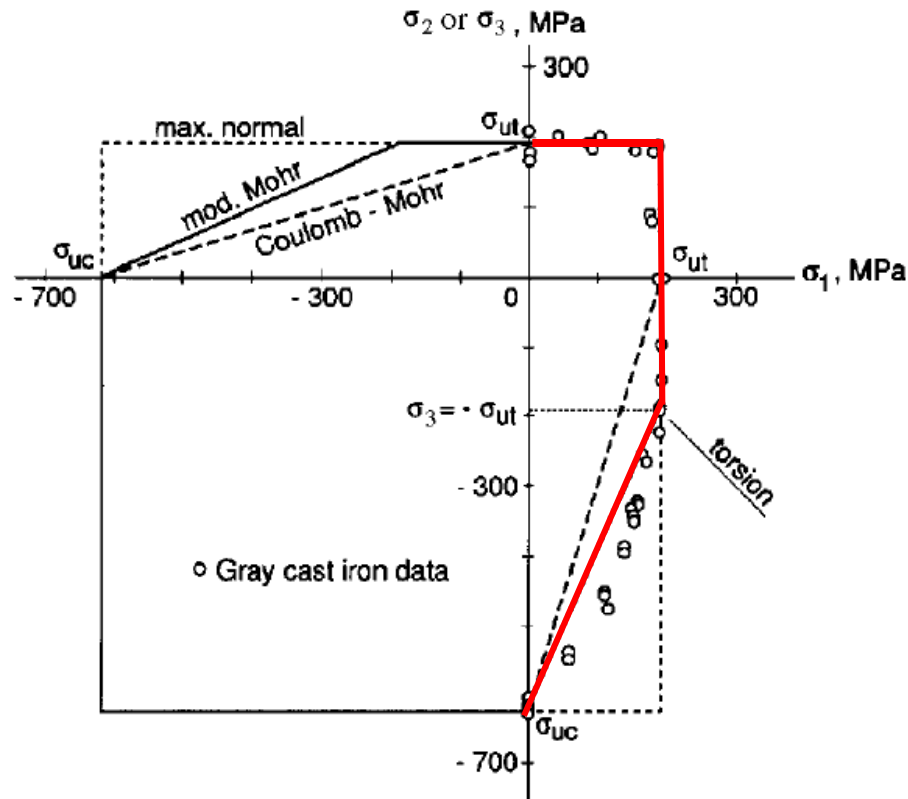


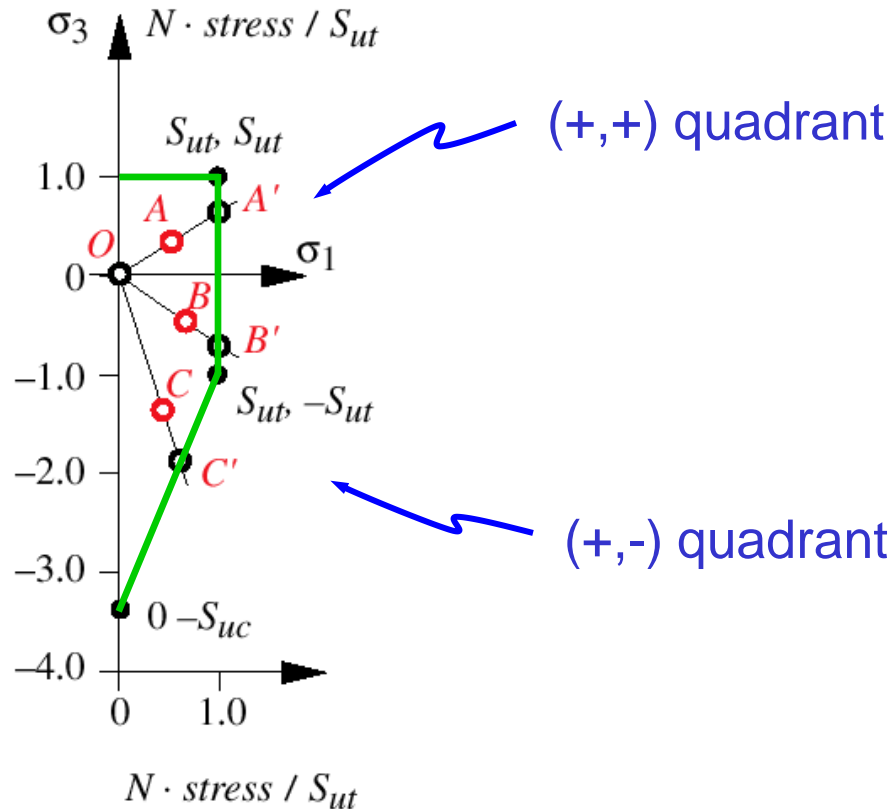
FIGURE 5-12

Biaxial Fracture Data of Gray Cast Iron Compared to Various Failure Criteria (From Fig 7.13, p. 255, in *Mechanical Behavior of Materials* by N. E. Dowling, Prentice-Hall, Englewood Cliffs, NJ, 1993. Data from R. C. Grassi and I. Cornet, "Fracture of Gray Cast Iron Tubes under Biaxial Stresses," *J. App. Mech*, v. 16, p.178, 1949)



# Static failure theories: brittle materials

## Modified-Mohr theory: quadrants of interest



**FIGURE 5-13**

Modified-Mohr Failure  
Theory for Brittle  
Material



# Static failure theories: brittle materials

## *Modified-Mohr theory*

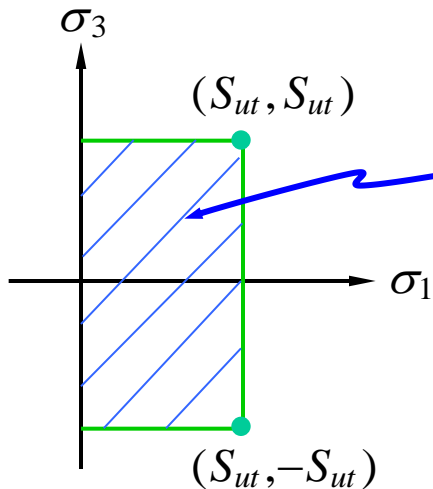
Safety factor: zone I:

Modified-Mohr theory:

$$SF = N = \frac{S_{ut}}{\sigma_1}$$

*Ultimate strength of the material in tension*

*Max. principal normal stress*



*Modified-Mohr theory: applicable inside this area*



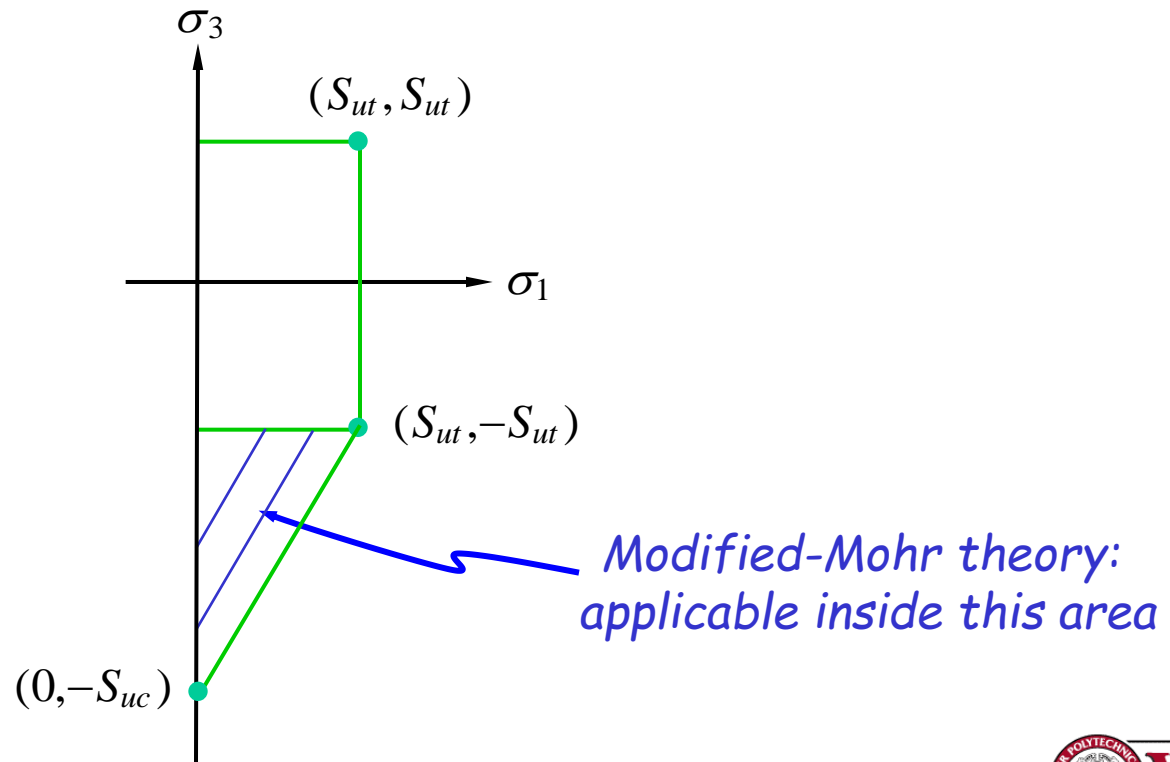
# Static failure theories: brittle materials

## *Modified-Mohr theory*

Safety factor: zone II

Modified-Mohr theory:

$$SF = N = \frac{S_{ut} |S_{uc}|}{|S_{uc}| \sigma_1 - S_{ut} (\sigma_1 + \sigma_3)}$$



# Static failure theories: brittle materials

## Modified-Mohr theory

Safety factor: zone II

Modified-Mohr theory:

$$SF = N = \frac{S_{ut} |S_{uc}|}{|S_{uc}| \sigma_1 - S_{ut} (\sigma_1 + \sigma_3)}$$

**EC: derive expression for the SF in Zone II**

Understand: state of stresses at points A, B, and C.

What do points A', B', and C' represent?

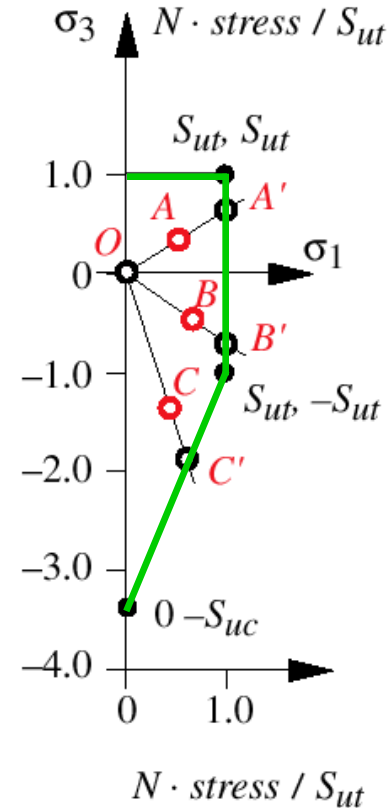


FIGURE 5-13

Modified-Mohr Failure Theory for Brittle Material



# Static failure theories: brittle materials

## *Effective stress: Dowling indexes*

(Similar concept as the equivalent von Mises stress in ductile materials)

$$C_1 = \frac{1}{2} \left[ |\sigma_1 - \sigma_2| + \frac{2S_{ut} - |S_{uc}|}{-|S_{uc}|} (\sigma_1 + \sigma_2) \right]$$

$$C_2 = \frac{1}{2} \left[ |\sigma_2 - \sigma_3| + \frac{2S_{ut} - |S_{uc}|}{-|S_{uc}|} (\sigma_2 + \sigma_3) \right]$$

$$C_3 = \frac{1}{2} \left[ |\sigma_1 - \sigma_3| + \frac{2S_{ut} - |S_{uc}|}{-|S_{uc}|} (\sigma_1 + \sigma_3) \right]$$



# Static failure theories

## Ductile materials

### Safety factors:

Distortion energy theory:

$$SF = N = \frac{S_y}{\sigma'}$$

Yield strength of the material

von Mises effective stress

(Obtained from)

Distortion energy theory (pure shear):

$$SF = N = \frac{S_{ys}}{\tau_{\max}}$$

$S_{ys} = 0.577S_y$

Max. shear-stress

Max. shear-stress theory:

$$SF = N = \frac{S_{ys}}{\tau_{\max}}$$

$S_{ys} = 0.5S_y$

Maximum shear-stress



# Static failure theories: brittle materials

## *Modified-Mohr theory: effective stress*

Safety factor:

Modified-Mohr theory.  
Effective stress:

$$SF = N = \frac{S_{ut}}{\tilde{\sigma}}$$

*Ultimate strength of the material in tension*

*Effective stress. Obtained as:*

$$\tilde{\sigma} = \text{MAX}(\sigma_1, \sigma_2, \sigma_3, C_1, C_2, C_3)$$

*and*

$$\tilde{\sigma} = 0 \quad \text{if } \text{MAX} < 0, \text{ use a different approach}$$



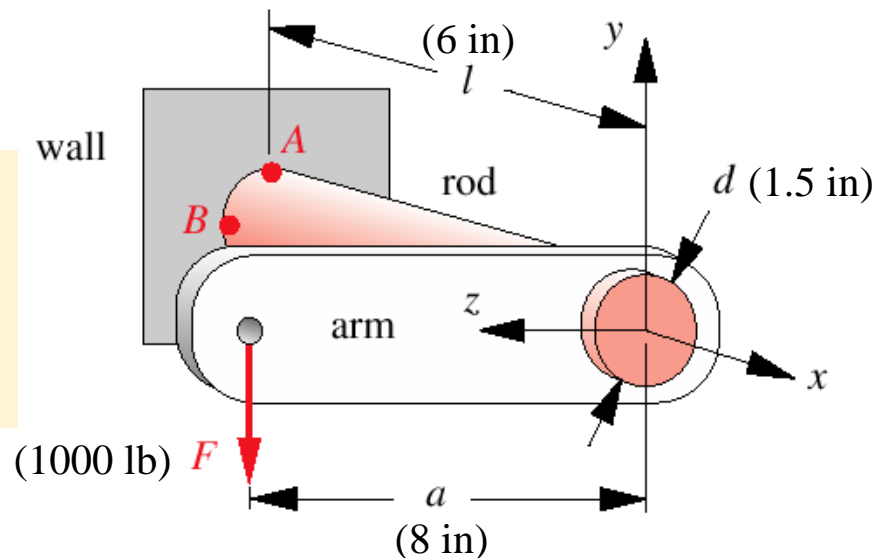


# Static failure theories: ductile & brittle materials

## Review and Master: Examples 5-1 and 5-2

Determine the safety factors for the bracket rod shown considering: (a) ductile; and (b) brittle materials.

Do analyses with & without stress concentrations at the wall/rod interface



Ductile case:

Al 2024-T4 (consult Appendix C)

$$S_y = 47 \text{ kpsi}$$

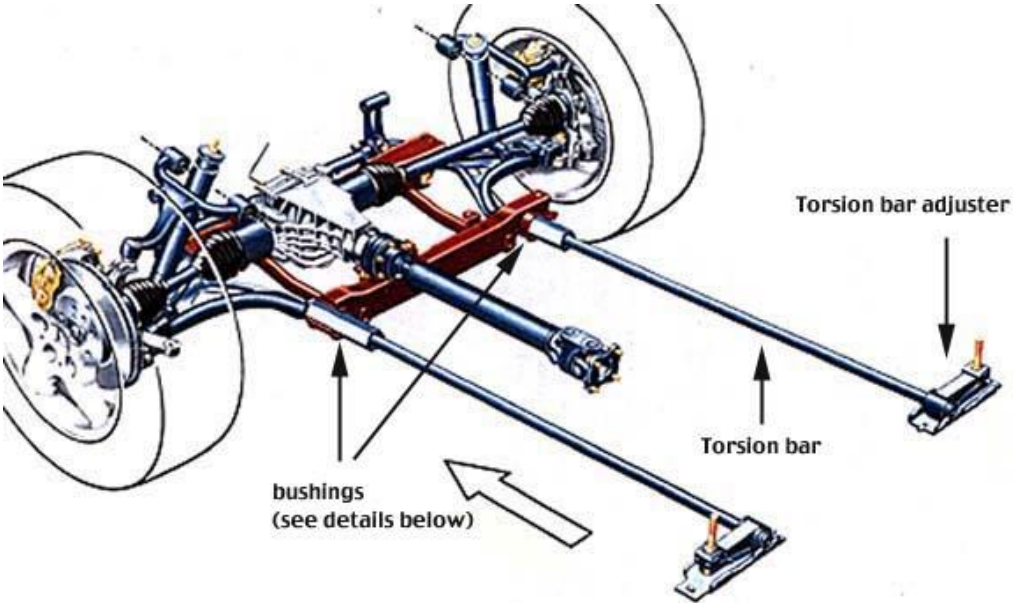
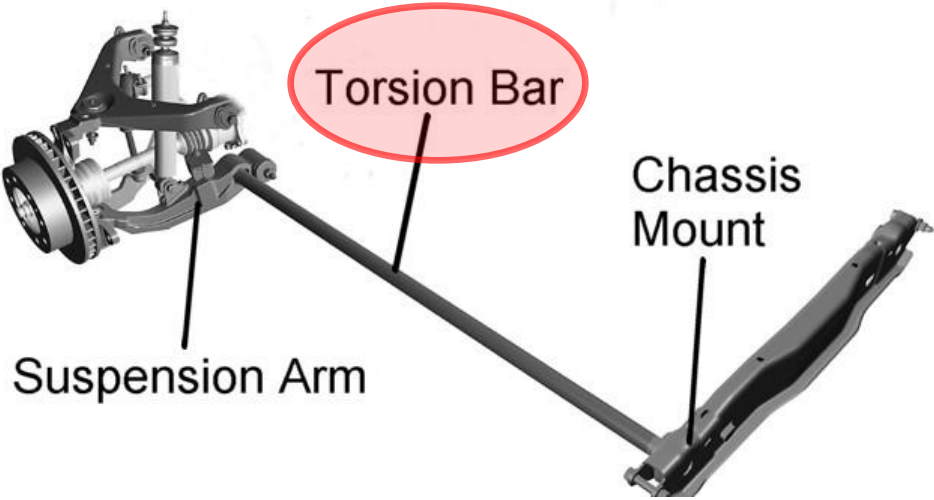
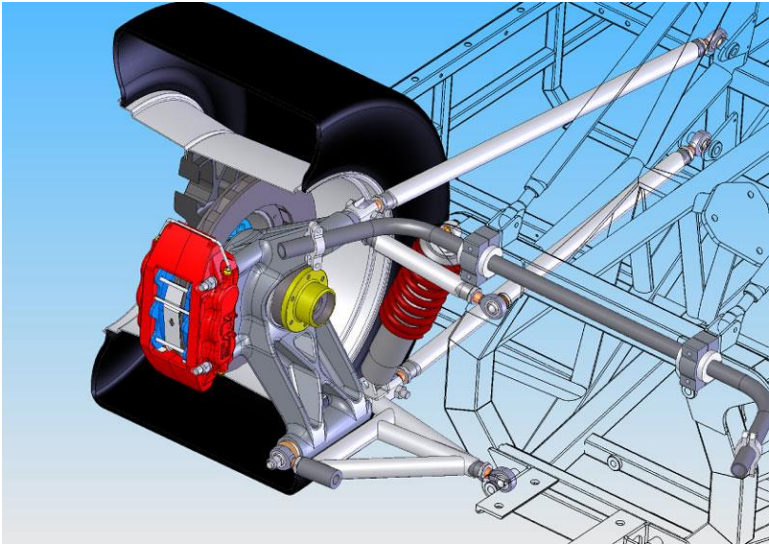
Brittle case:

Class 50 gray cast iron (consult Appendix C)

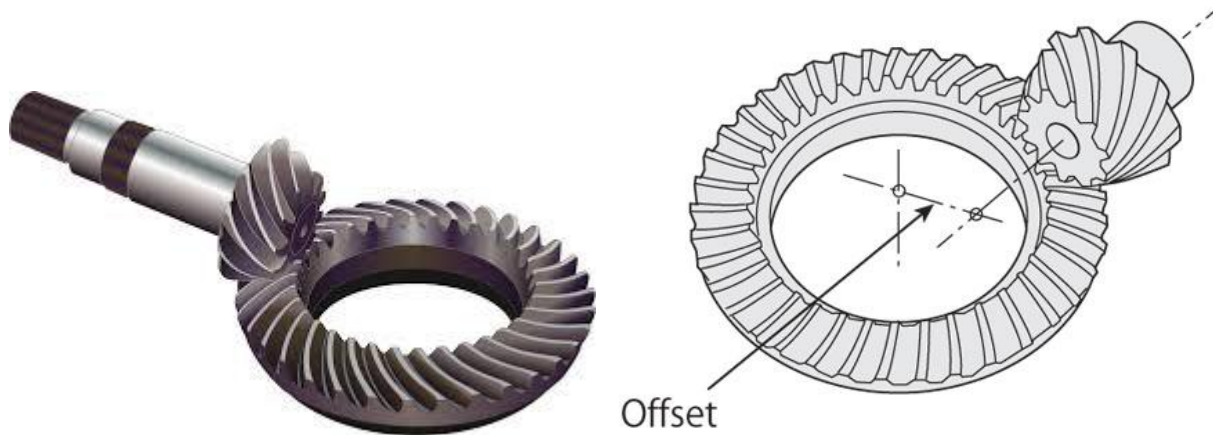
$$S_{ut} = 52.5 \text{ kpsi. } S_{uc} = 164 \text{ kpsi}$$



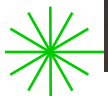
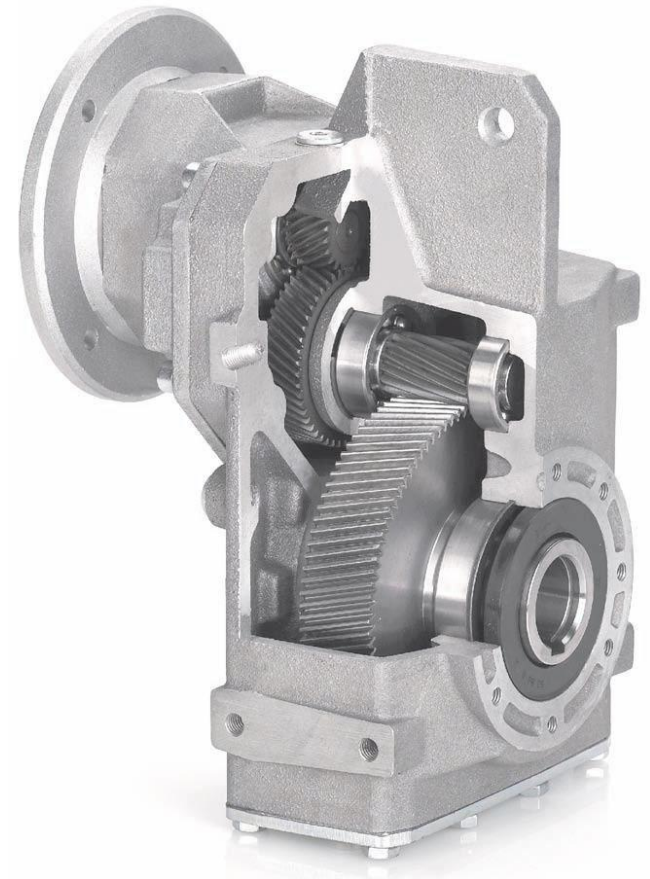
# Uses of the bracket model configuration: suspension system



# Uses of the bracket model configuration: transmissions



Hypoid Gear



# Static failure theories

## Ductile materials

### Safety factors:

Distortion energy theory:

$$SF = N = \frac{S_y}{\sigma'}$$

Yield strength of the material

von Mises effective stress

(Obtained from)

Distortion energy theory (pure shear):

$$SF = N = \frac{S_{ys}}{\tau_{\max}}$$

$S_{ys} = 0.577S_y$

Max. shear-stress

Max. shear-stress theory:

$$SF = N = \frac{S_{ys}}{\tau_{\max}}$$

$S_{ys} = 0.5S_y$

Maximum shear-stress



# Static failure theories: brittle materials

## *Modified-Mohr theory: effective stress*

Safety factor:

Modified-Mohr theory.  
Effective stress:

$$SF = N = \frac{S_{ut}}{\tilde{\sigma}}$$

Ultimate strength of the material in tension

Effective stress. Obtained as:

$$\tilde{\sigma} = \text{MAX}(\sigma_1, \sigma_2, \sigma_3, C_1, C_2, C_3)$$

and

$$\tilde{\sigma} = 0 \quad \text{if } \text{MAX} < 0, \text{ use a different approach}$$



# Review Example

A circular rod is subjected to combined loading consisting of a tensile load  $P = 10$  kN and a torque  $T = 5$  kN·m. Rod is 50 mm in diameter.

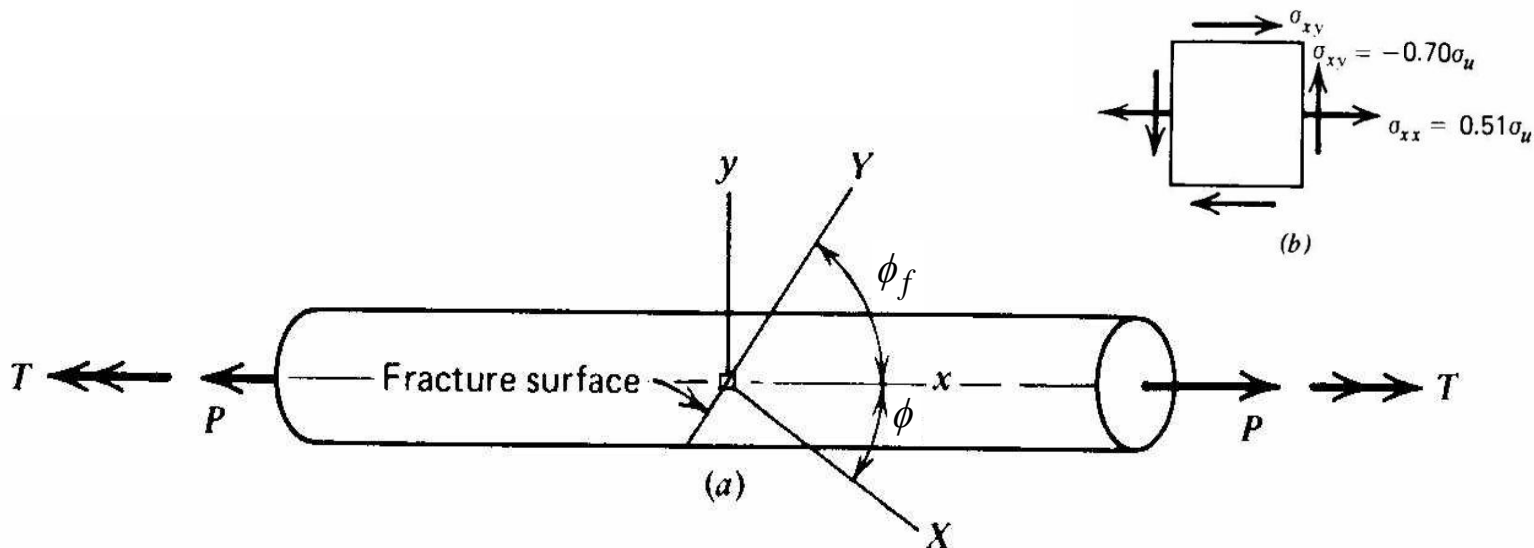
- 1) Draw stress element (cube) at the most highly stressed location on the rod, and
- 2) draw corresponding Mohr's circle(s).



# Review Example

A piece of chalk is subjected to combined loading consisting of a tensile load  $P$  and a torque  $T$ , see figure. The chalk has an ultimate strength  $\sigma_u$  as determined by a tensile test. The load  $P$  remains constant at such a value that it produces a tensile stress of  $0.51 \cdot \sigma_u$  on any cross-section. The torque  $T$  is increased gradually until fracture occurs on some inclined surface.

Assuming that fracture takes place when the maximum principal stress  $\sigma_1$  reaches the ultimate strength  $\sigma_u$ , **determine the magnitude of the torsional shearing stress produced by the torque  $T$  at fracture and determine the orientation of the fractured surface.**



# Reading assignment

- Chapters 5 of textbook: Sections 5.2 to 5.5
- Review notes and text: ES2501, ES2502

# Homework assignment

- **Author's:** As indicated in website of our course
- **Solve:** As indicated in website of our course

