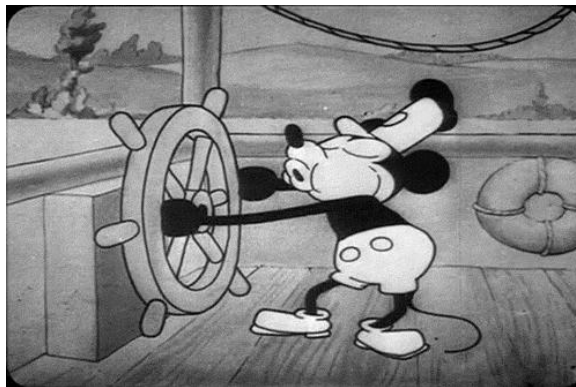


WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

STRESS ANALYSIS ES-2502, B'2025

We will get started soon...



09 December 2025



WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

STRESS ANALYSIS ES-2502, B'2025

Lecture 25:

Unit 27:
Combined loading (Ch. 8)

09 December 2025



General information

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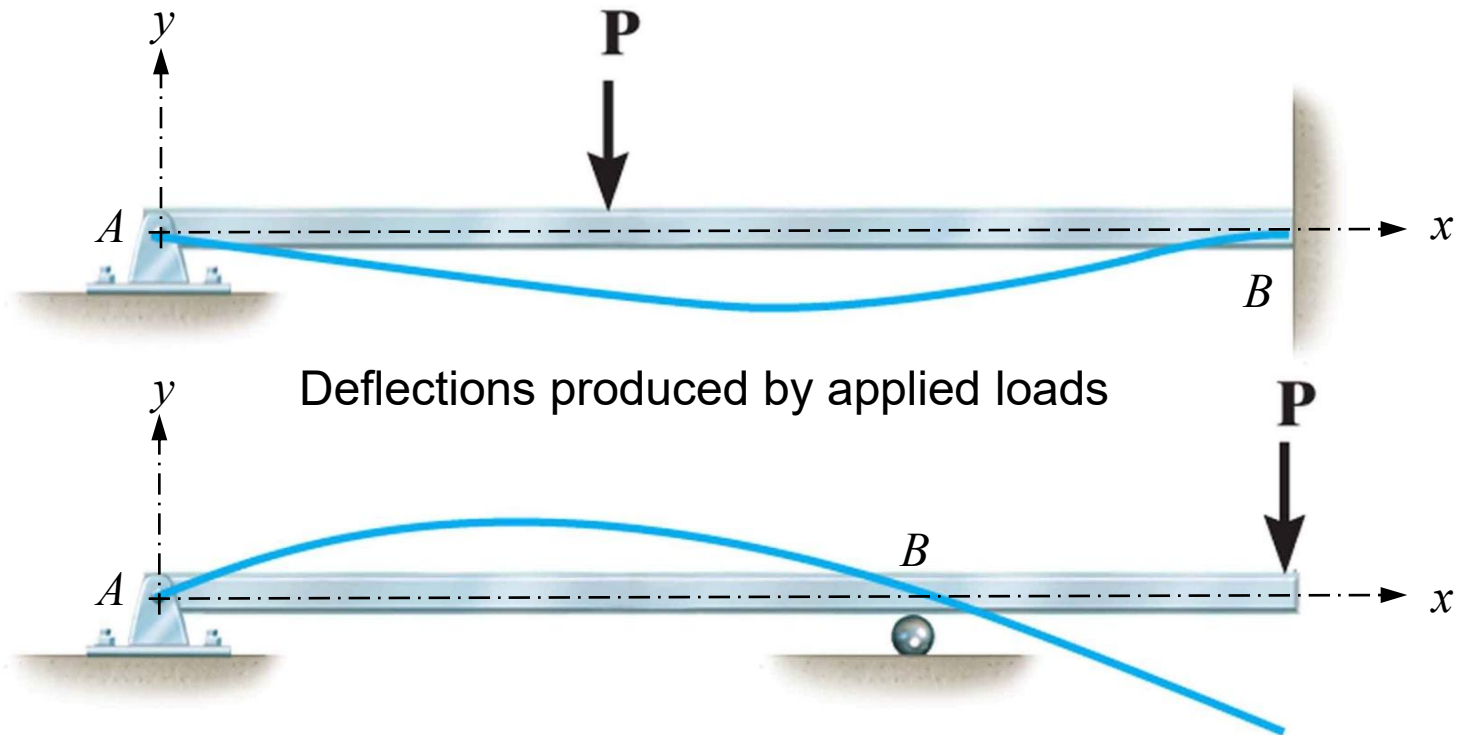


Deflection of beams and shafts

The elastic curve

We can study how beams and shafts deflect by knowing **both**:

- a) Distribution of bending moments (V - M diagrams), and
- b) Material & geometrical properties of components



Bending deformation of straight beams

The elastic curve

Radius of curvature is
computed by:

$$\frac{1}{\rho} = \frac{\frac{d^2 y}{dx^2}}{\left[1 + \left(\frac{dy}{dx} \right)^2 \right]^{2/3}}$$

Therefore,

$$\frac{\frac{d^2 y}{dx^2}}{\left[1 + \left(\frac{dy}{dx} \right)^2 \right]^{2/3}} = \frac{M}{E \cdot I_{zz}}$$



Bending deformation of straight beams

The elastic curve

For small
deformations:

$$\frac{d^2 y}{dx^2} = \frac{M}{E \cdot I_{zz}}$$

← *Elastica*
equation

*Important to
remember!!*



Bending deformation of straight beams

The elastic curve

For small
deformations:

$$\frac{d^2 y}{dx^2} = \frac{M}{E \cdot I_{zz}}$$

← *Elastica*
equation

$$\frac{d}{dx} \left(E \cdot I_{zz} \frac{d^2 y}{dx^2} \right) = V(x)$$

Shear force

$$\frac{d^2}{dx^2} \left(E \cdot I_{zz} \frac{d^2 y}{dx^2} \right) = w(x)$$

Applied load



Bending deformation of straight beams

The elastic curve

$$E \cdot I_{zz} \frac{d^4 y}{dx^4} = w(x) \quad \text{Applied load}$$

$$E \cdot I_{zz} \frac{d^3 y}{dx^3} = V(x) \quad \text{Shear force}$$

$$E \cdot I_{zz} \frac{d^2 y}{dx^2} = M \quad \leftarrow \text{Elastica equation}$$



Bending deformation of straight beams

The elastic curve

$$\frac{w}{EI} = \frac{d^4 y}{dx^4}$$

Load function – deflection

$$\frac{V}{EI} = \frac{d^3 y}{dx^3}$$

Shear function – deflection

$$\frac{M}{EI} = \frac{d^2 y}{dx^2}$$

Moment function – *elastica*

$$\theta = \frac{dy}{dx}$$

Slope – deflection

$$y = f(x)$$

Deflection

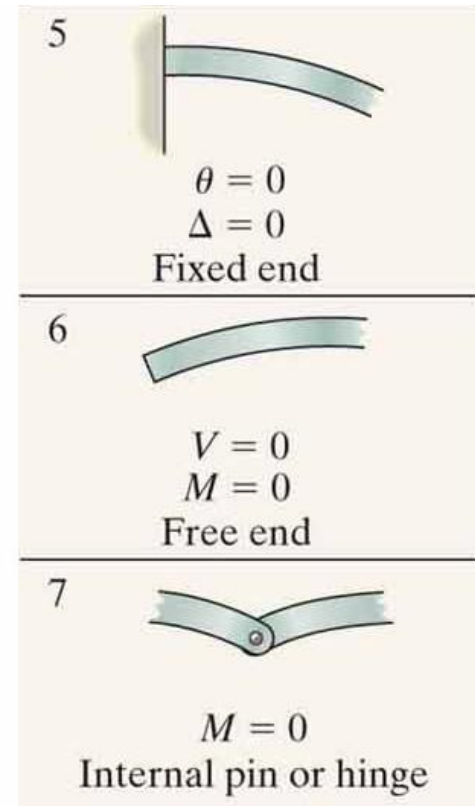
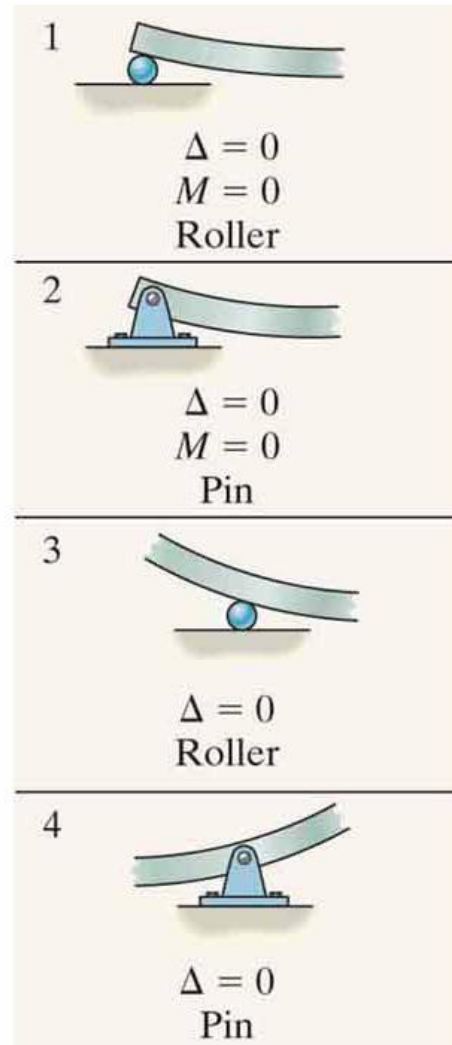


Bending deformation of straight beams

Boundary and continuity conditions

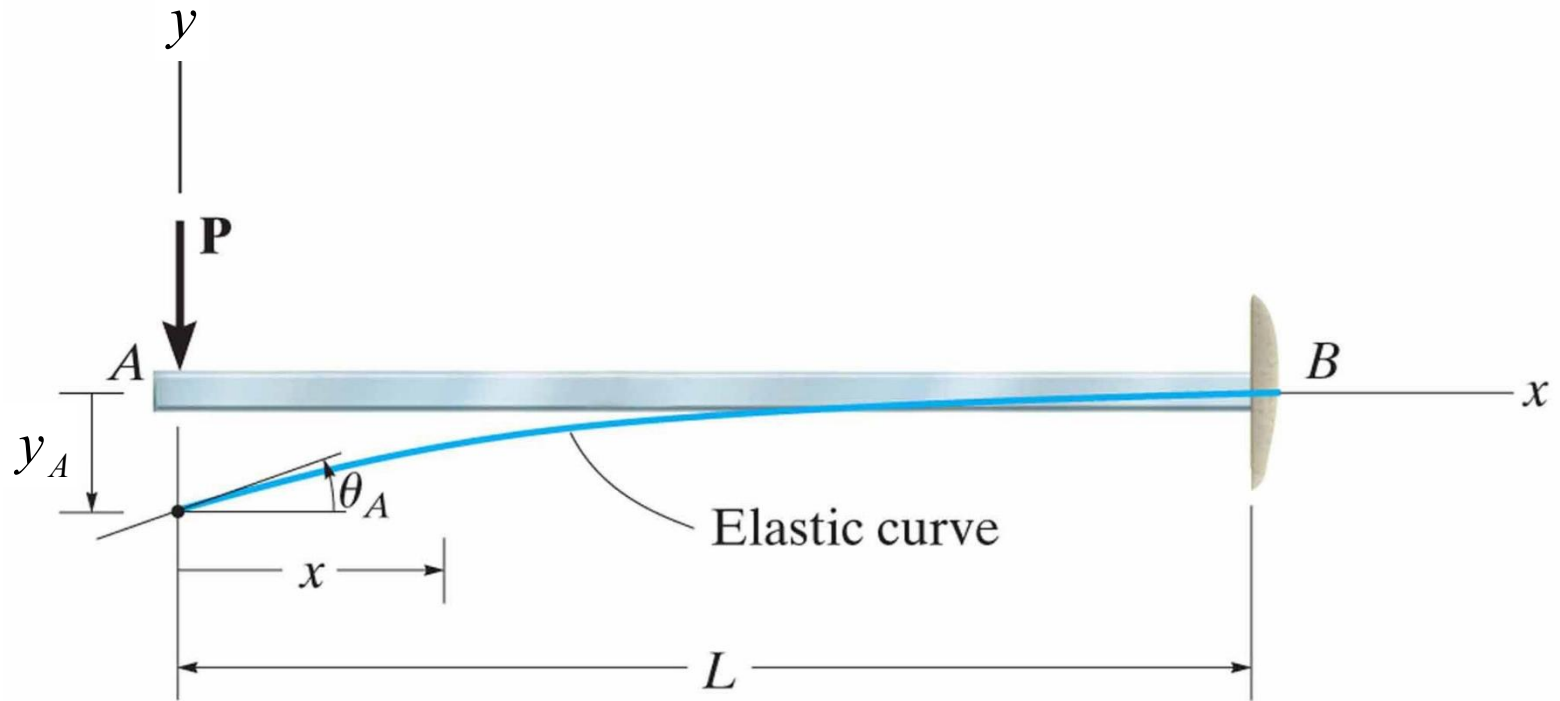
Δ = displacement

θ = slope of
displacement



Bending deformation of straight beams: example A

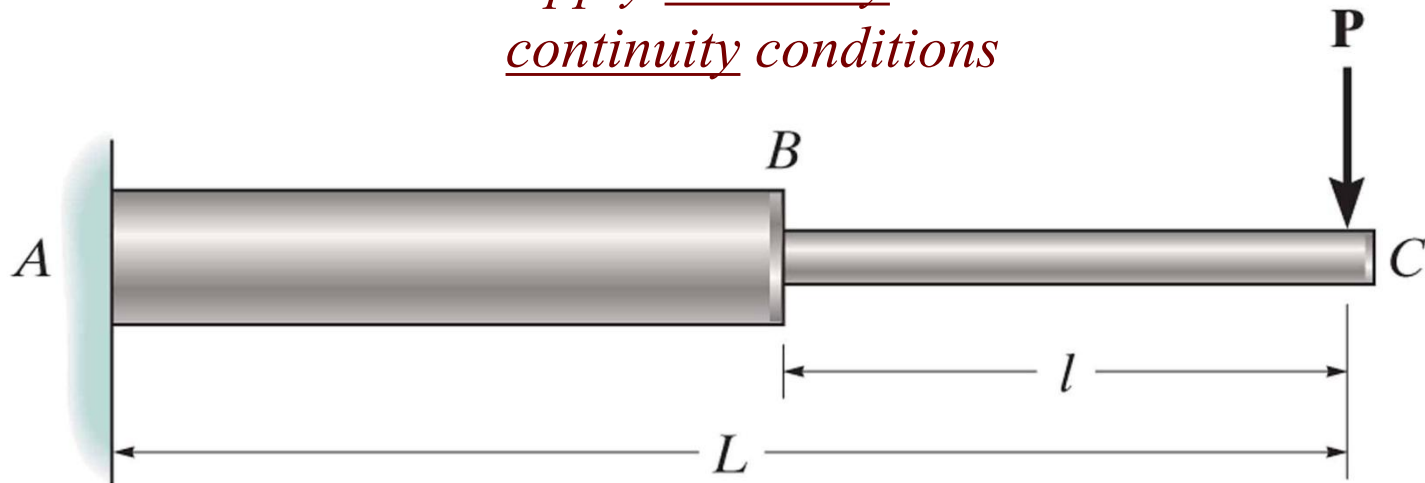
The cantilever shown is subjected to a vertical load P at its end. Determine the equation of the deformation (elastic) curve. $E \cdot I$ is constant.



Bending deformation of straight beams: example B

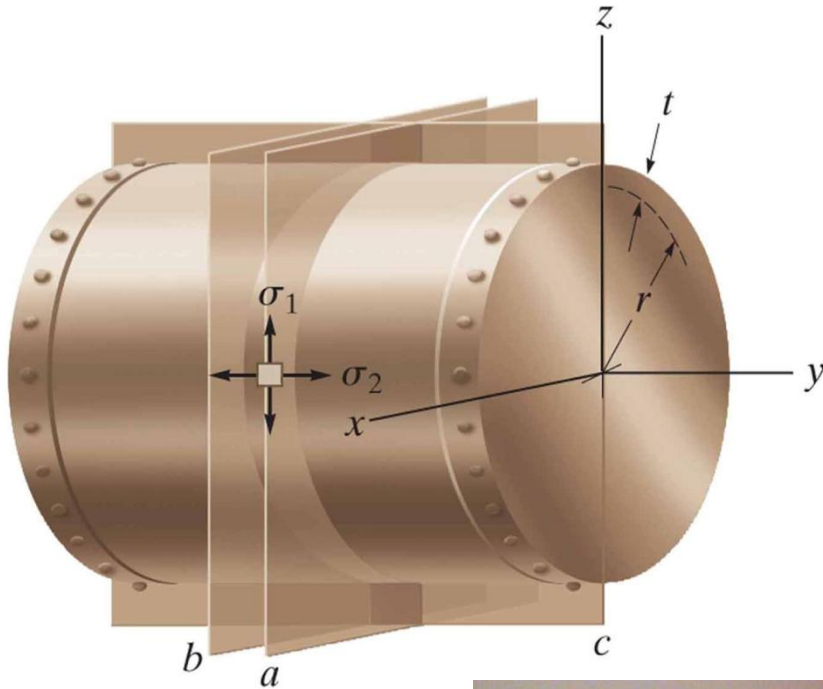
The beam is made of two rods and is subjected to the concentrated load P . Determine the maximum deflection of the beam if the moments of inertia of the rods are I_{AB} and I_{BC} , and the modulus of elasticity is E .

Apply boundary and continuity conditions



Combined loading

Thin-wall vessels: 2D state of stresses



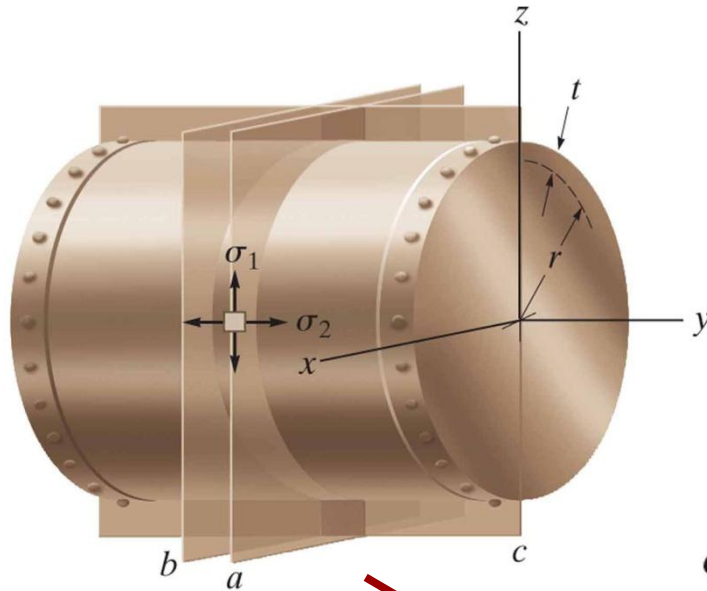
Thin-wall condition:

$$\frac{r}{t} \geq 10$$



Combined loading

Thin-wall vessels: 2D state of stresses

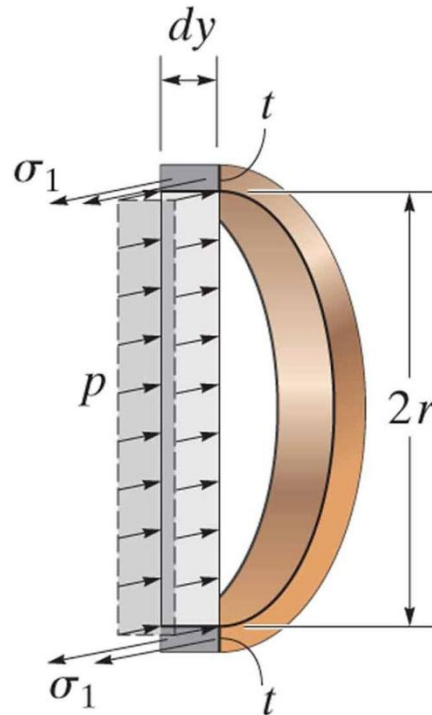


$$\sum F_x = 0 \Rightarrow$$

$$2 \sigma_1 (t \cdot dy) - p (2 r dy) = 0$$

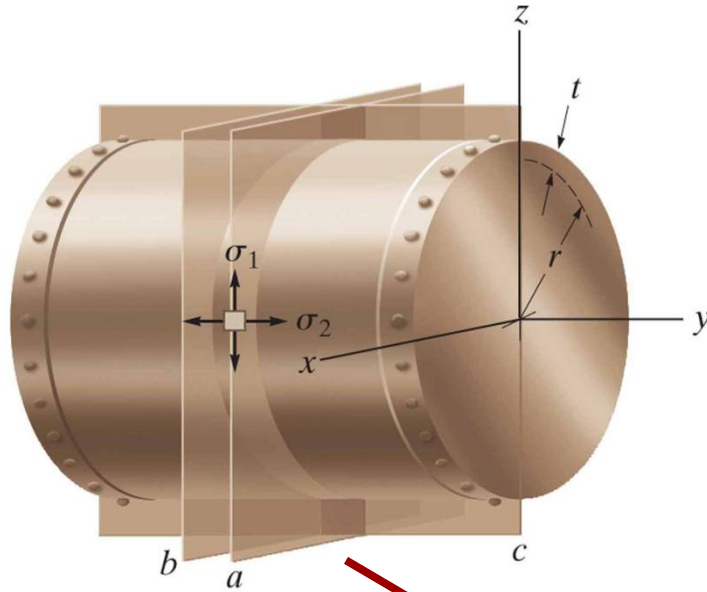
$$\Rightarrow \sigma_1 = \frac{P \cdot r}{t}$$

*Tangential
component of
stresses*



Combined loading

Thin-wall vessels: 2D state of stresses

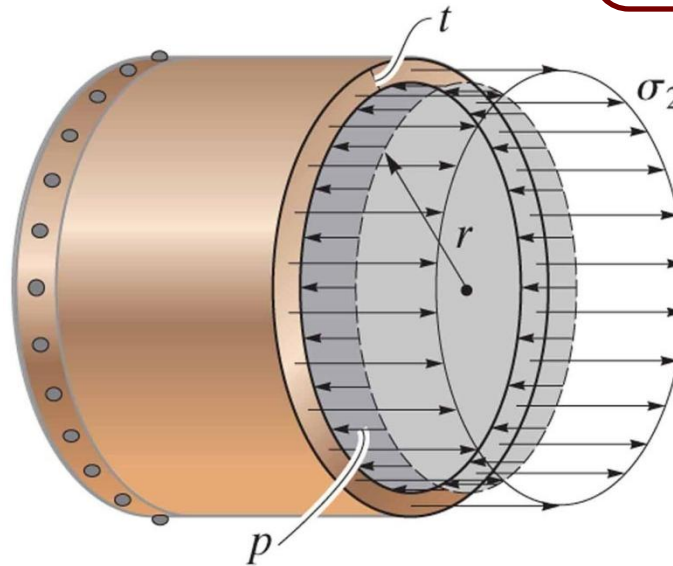


$$\sum F_y = 0 \Rightarrow$$

$$\sigma_2(2\pi \cdot r \cdot t) - p(\pi \cdot r^2) = 0$$

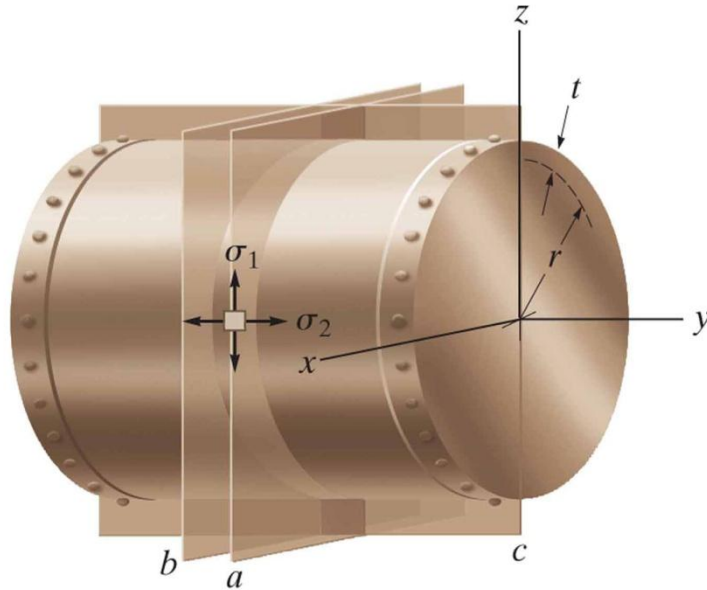
$$\Rightarrow \sigma_2 = \frac{P \cdot r}{2t}$$

*Longitudinal
component of
stresses*



Combined loading

Thin-wall vessels: 2D state of stresses

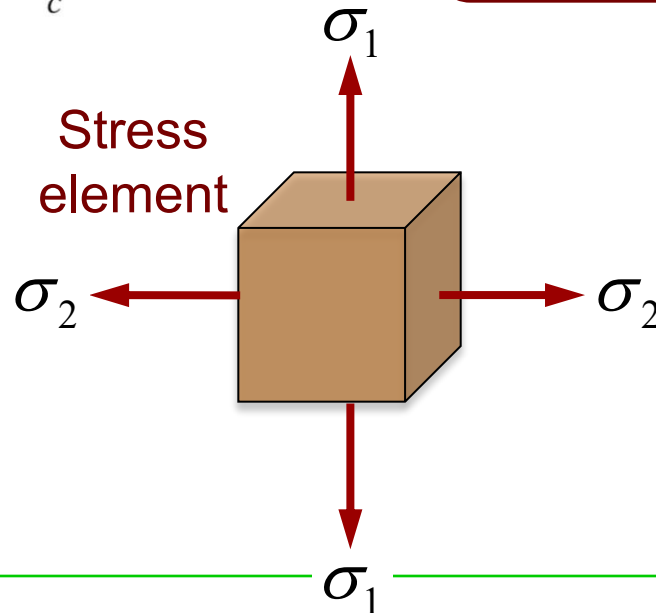


$$\sigma_1 = \frac{P \cdot r}{t}$$

*Radial
component of
stresses*

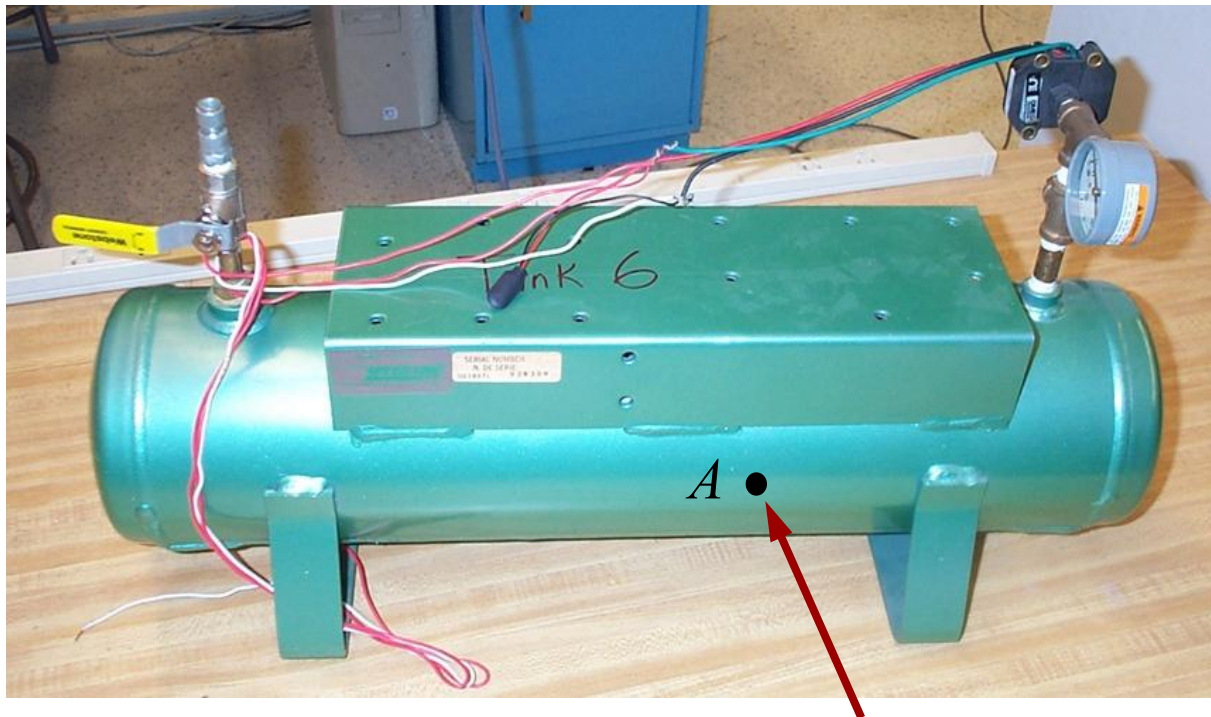
$$\sigma_2 = \frac{P \cdot r}{2t}$$

*Longitudinal
component of
stresses*



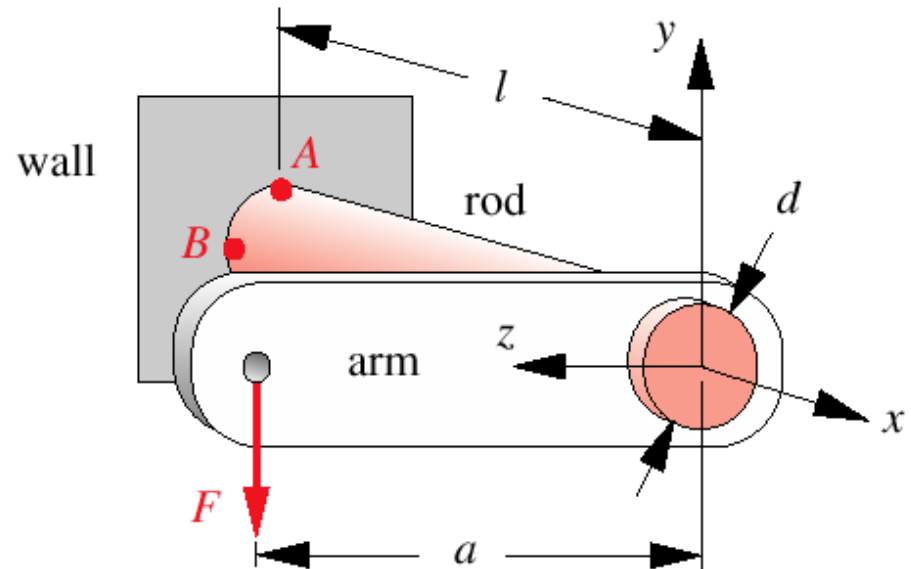
Combined loading: thin-wall vessels: example A

The tank of an air compressor is subjected to an internal pressure of 90 psi. If the internal diameter of the tank is 6.0 in, and the wall thickness is 0.10 in, determine the stress components acting at point *A*. Draw a volume (stress) element of the material at this point, and show the results on the element.

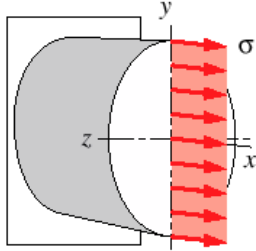


Combined loading

Find the most highly stressed locations on the bracket shown. Draw volume (stress) elements at points *A* and *B*

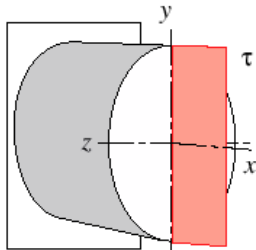


(a) Uniaxial tension, stress distribution across section



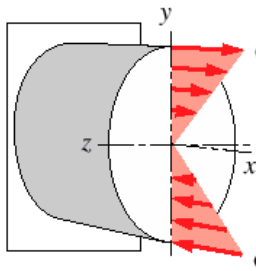
$$\sigma = \frac{P}{A}$$

(b) Direct shear, average-stress distribution across section



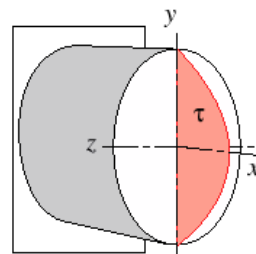
$$\tau = \frac{P}{A_{shear}}$$

(c) Bending, normal-stress distribution across section



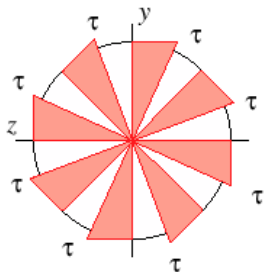
$$\sigma = \frac{My}{I}$$

(d) Bending, shear-stress distribution across section



$$\tau = \frac{VQ}{Ib}$$

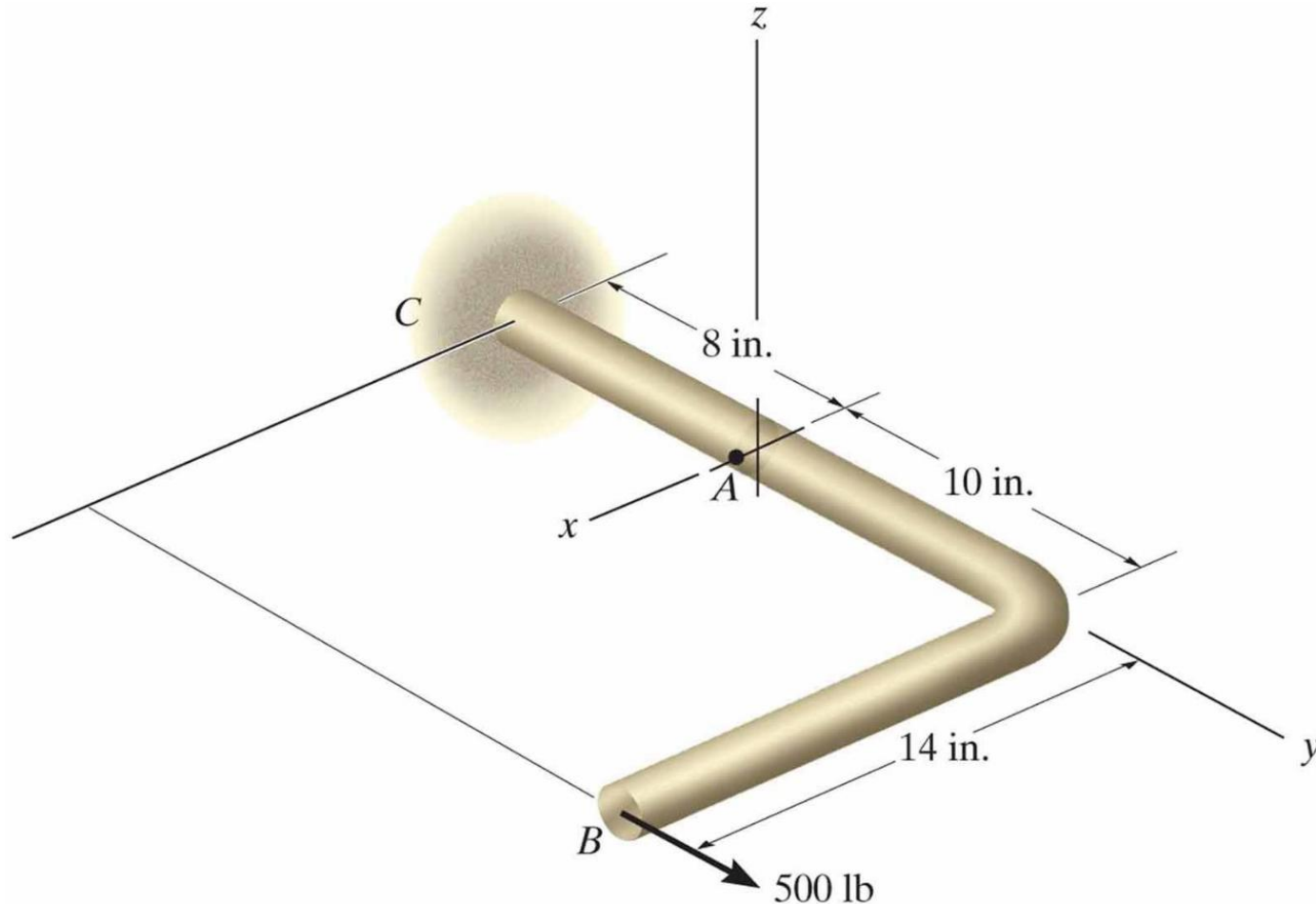
(e) Torsion, shear-stress distribution across section



$$\tau = \frac{Tr}{J}$$

Combined loading: example B

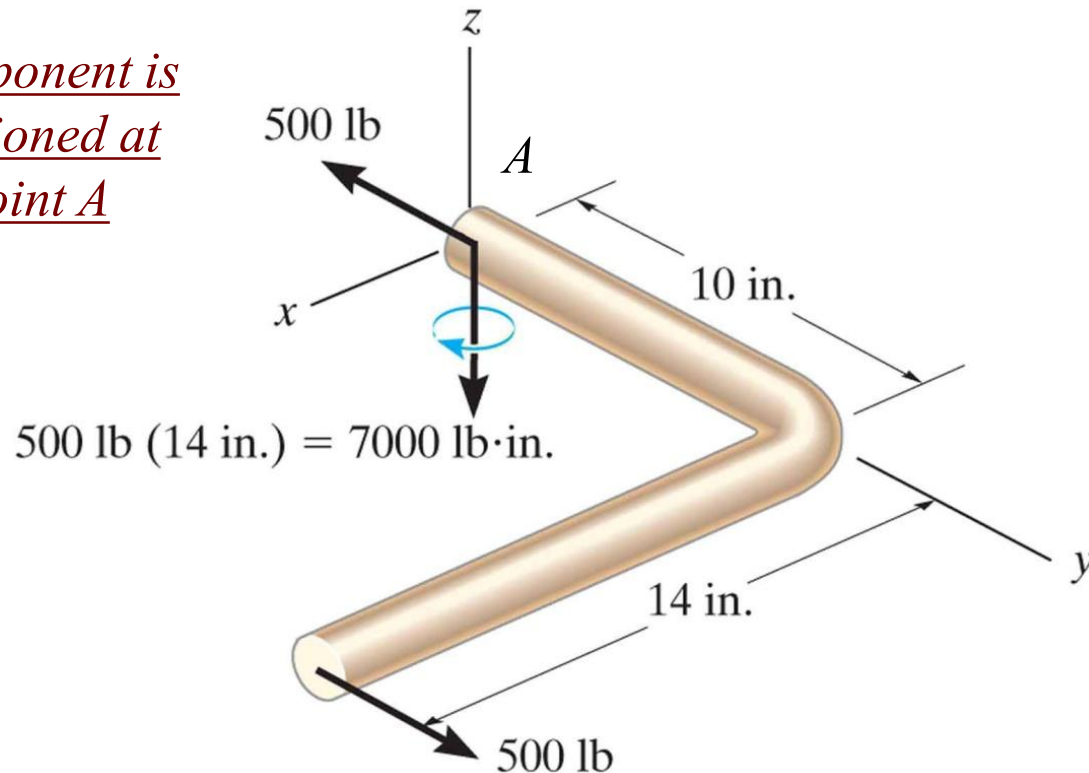
The solid rod shown has a radius of 0.75 in. If it is subjected to the force of 500 lbf, determine the state of stress at point A .



Combined loading: example B

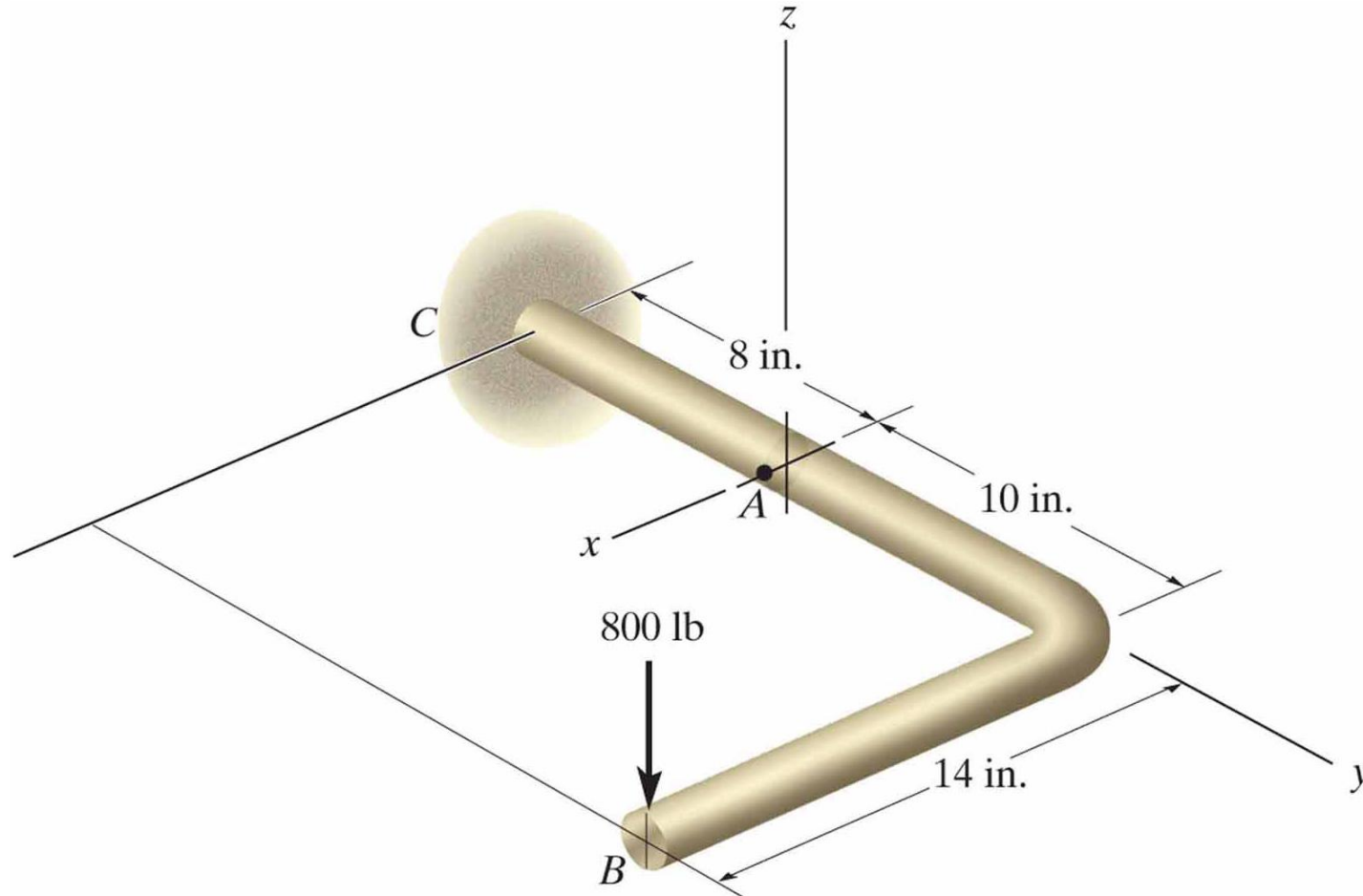
The solid rod shown has a radius of 0.75 in. If it is subjected to the force of 500 lbf, determine the state of stress at point *A*.

Component is
sectioned at
point *A*



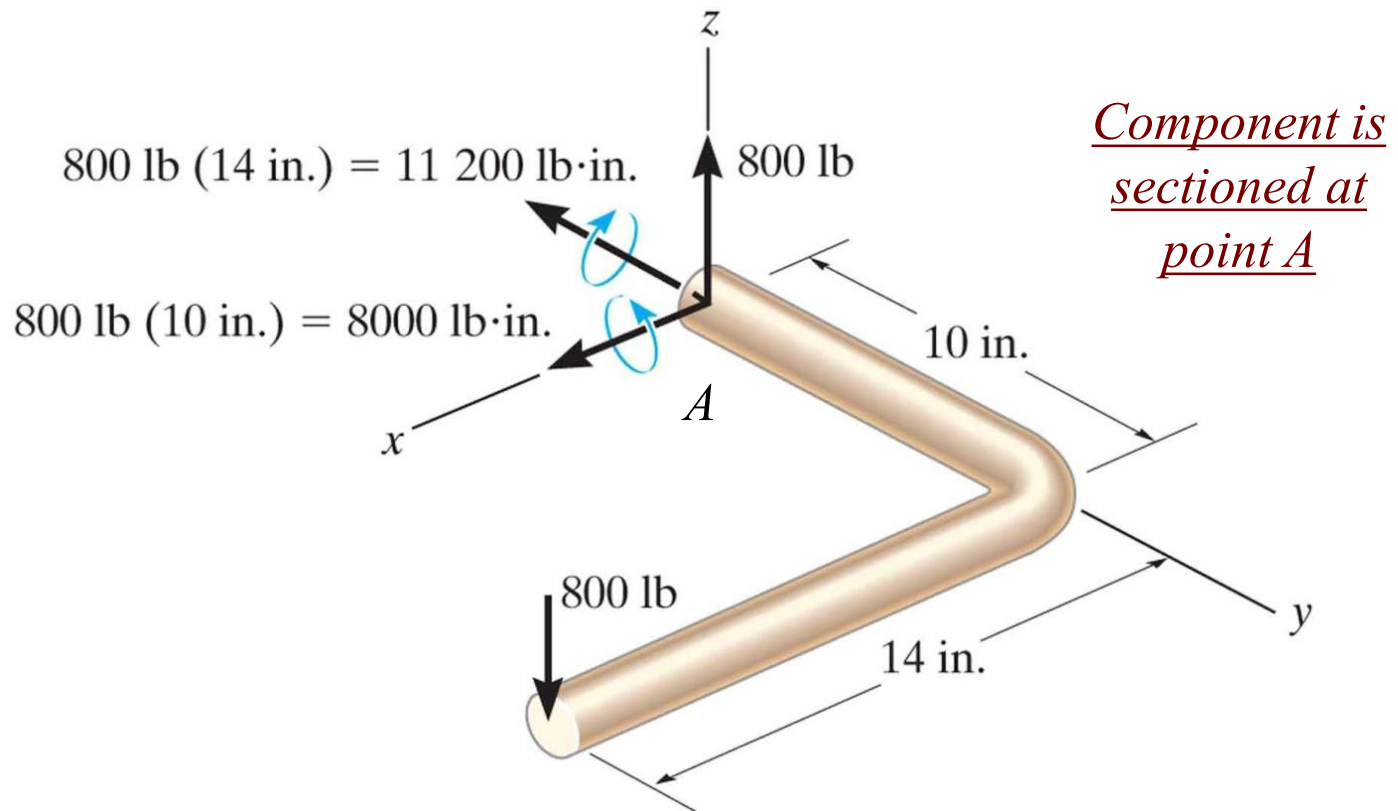
Combined loading: example C

The solid rod shown has a radius of 0.75 in. If it is subjected to the force of 800 lbf, determine the state of stress at point A .



Combined loading: example C

The solid rod shown has a radius of 0.75 in. If it is subjected to the force of 800 lbf, determine the state of stress at point *A*.



Reading assignment

- Chapters 8 and 12 of textbook
- Review notes and text: ES2001, ES2501



Homework assignment

- As indicated on webpage of our course

