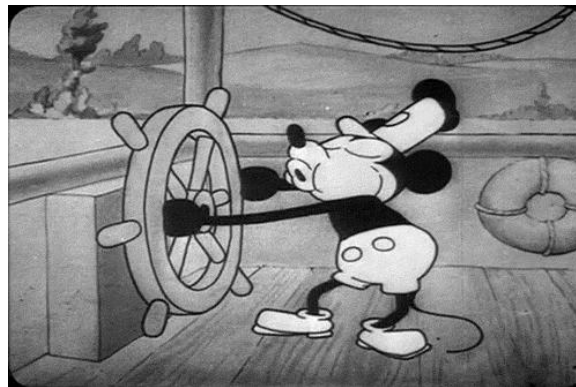


# WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

## STRESS ANALYSIS ES-2502, B'2025

We will get started soon...



02 December 2025



# WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

## STRESS ANALYSIS ES-2502, B'2025

Lecture 21:  
Unit 17: Bending of beams::  
*MV diagrams & MV general relationship*

02 December 2025



# General information

Instructor: Cosme Furlong

HL-152

(508) 831-5126

Email: cfurlong @ wpi.edu

<http://www.wpi.edu/~cfurlong/es2502.html>

## Graduate Assistants:

→ Hamed Ghavami (TA)

Email: sghavami @ wpi.edu

→ Jay Patil (GA)

Email: jpatil1 @ wpi.edu

→ Mikayla Almeida (GA)

mpalmeida @ wpi.edu

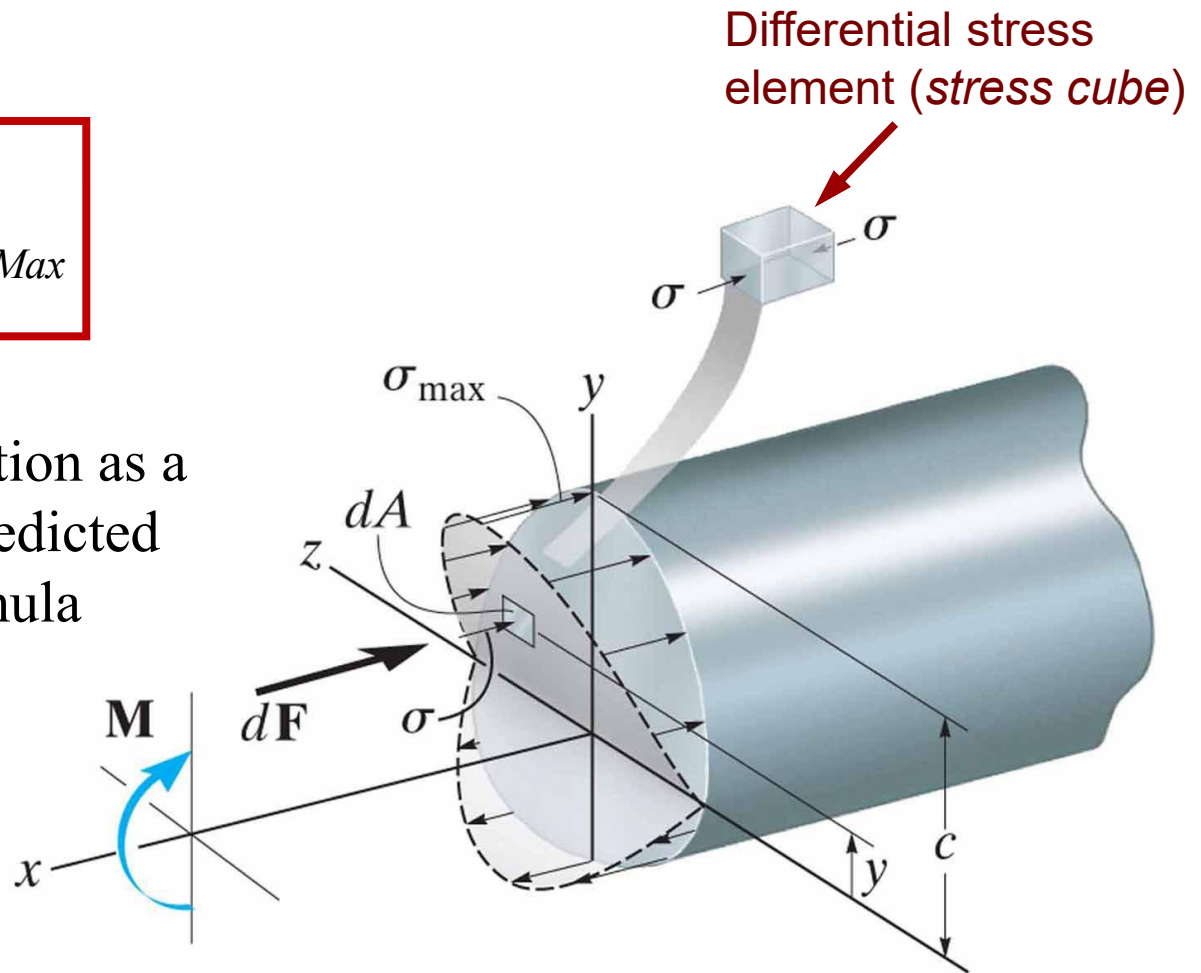


# The flexure formula

$$\sigma_x = \left( \frac{y}{c} \right) \cdot \sigma_{x \text{ Max}}$$

Normal stress variation as a function of  $y$  as predicted by flexure formula

$$\sigma_x = -E \frac{y}{\rho}$$
$$\sigma_{x \text{ Max}} = -E \frac{c}{\rho}$$

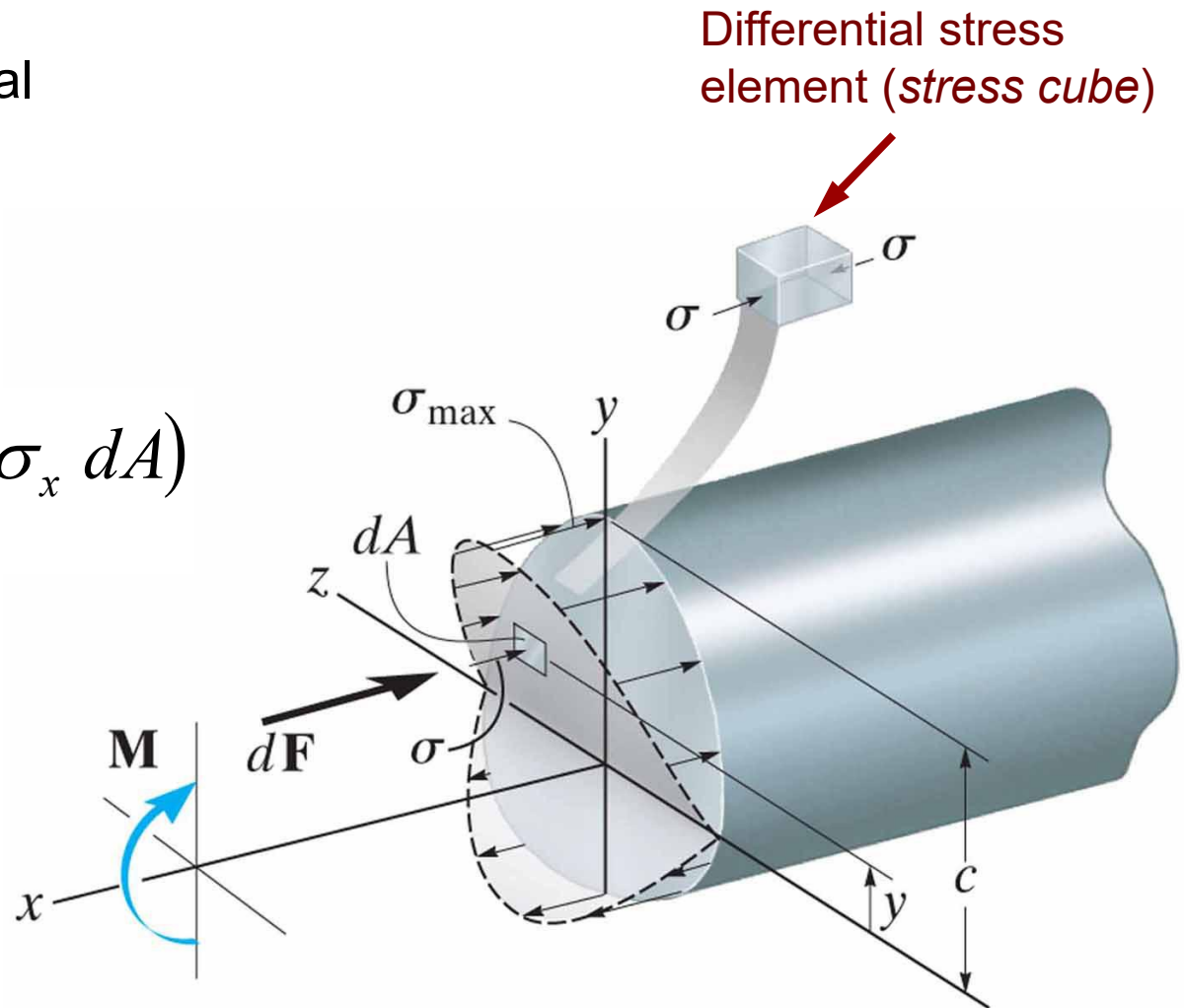


# The flexure formula

Resultant internal  
moment:

$$M = \sum M_z$$

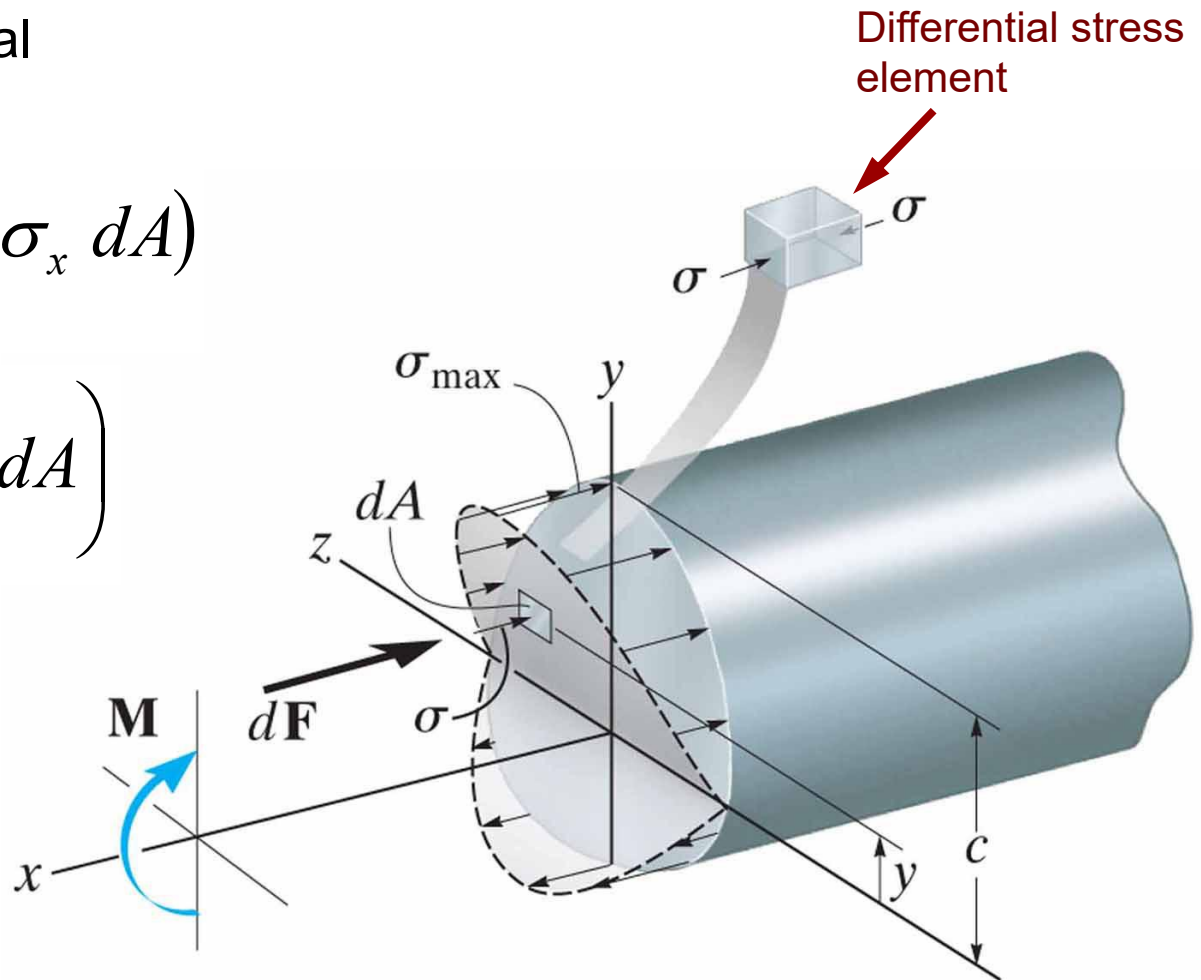
$$M = \int_A y \, dF = \int_A y (\sigma_x \, dA)$$



# The flexure formula

Resultant internal  
moment:

$$M = \int_A y \, dF = \int_A y (\sigma_x \, dA)$$
$$= \int_A y \left( \frac{y}{c} \sigma_{x_{Max}} \, dA \right)$$



# The flexure formula

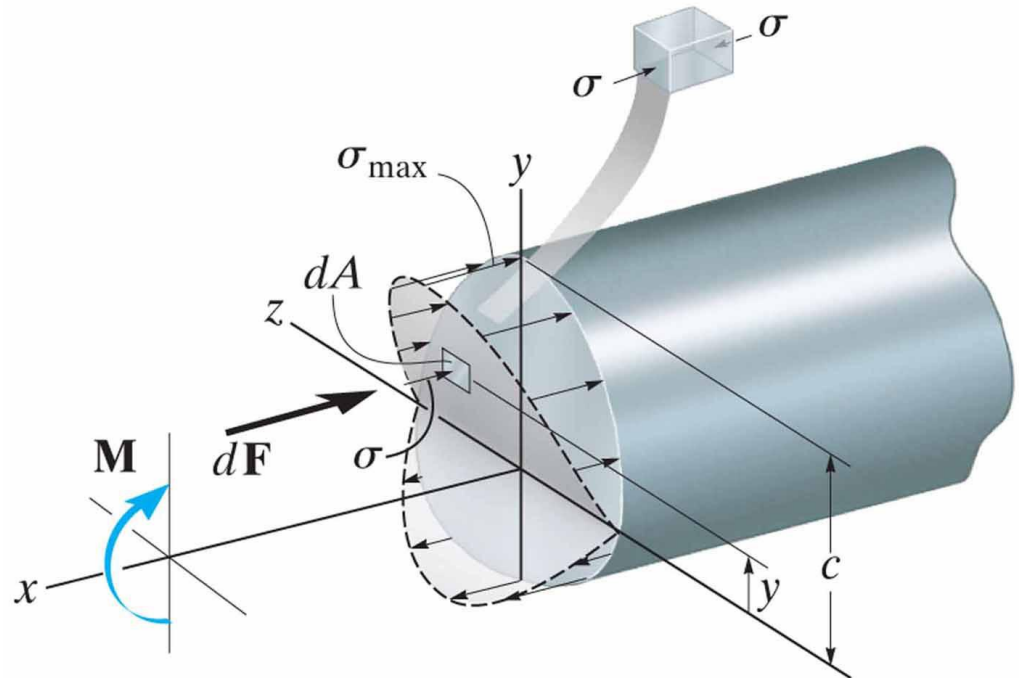
Resultant internal  
moment:

$$M = \frac{\sigma_{x_{Max}}}{c} \int_A y^2 dA$$

$$\sigma_{x_{Max}} = \frac{M c}{I_{zz}}$$

$$I_{zz} = \int_A y^2 dA$$

Area moment of  
inertia wrt to  $z$ -axis



# The flexure formula

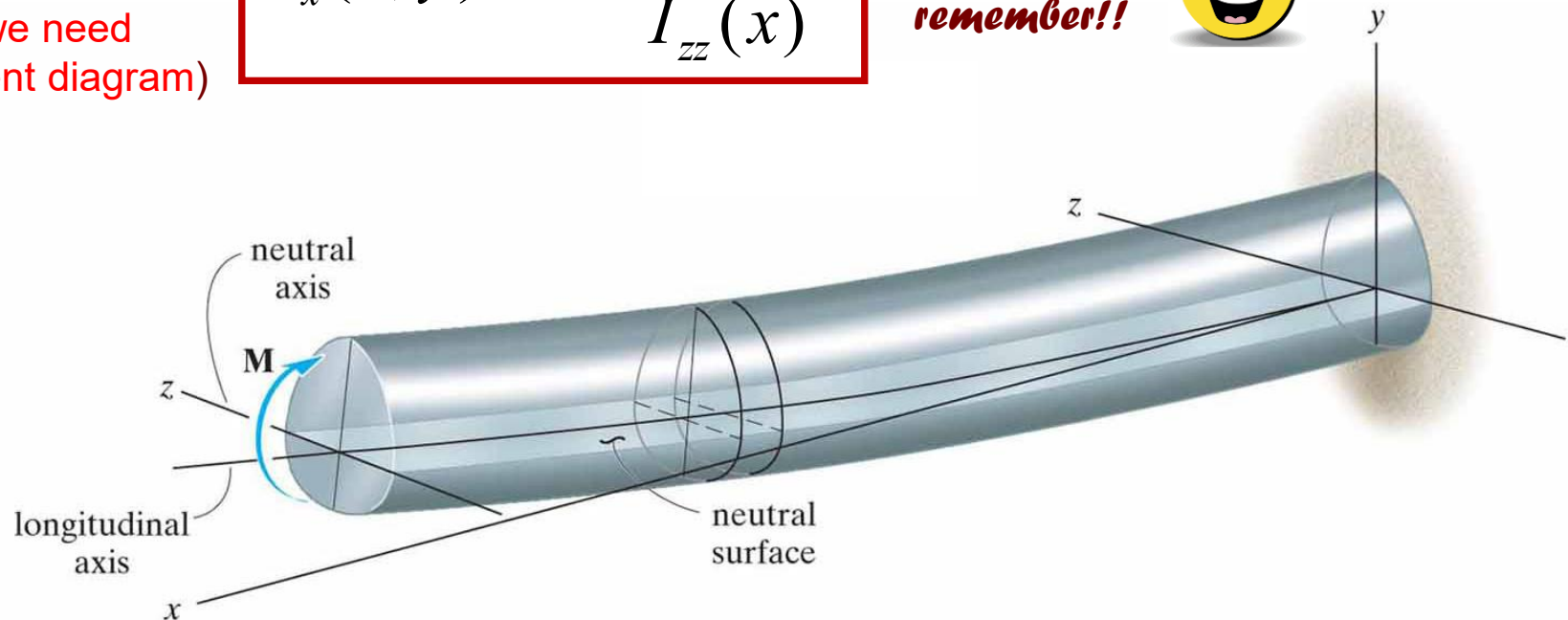
$$\sigma_{x_{Max}} = \frac{M c}{I_{zz}}$$

$$\sigma_x = -\frac{M y}{I_{zz}}$$

Do note that:  
(we need  
moment diagram)

$$\sigma_x(x, y) = -\frac{M(x) \cdot y}{I_{zz}(x)}$$

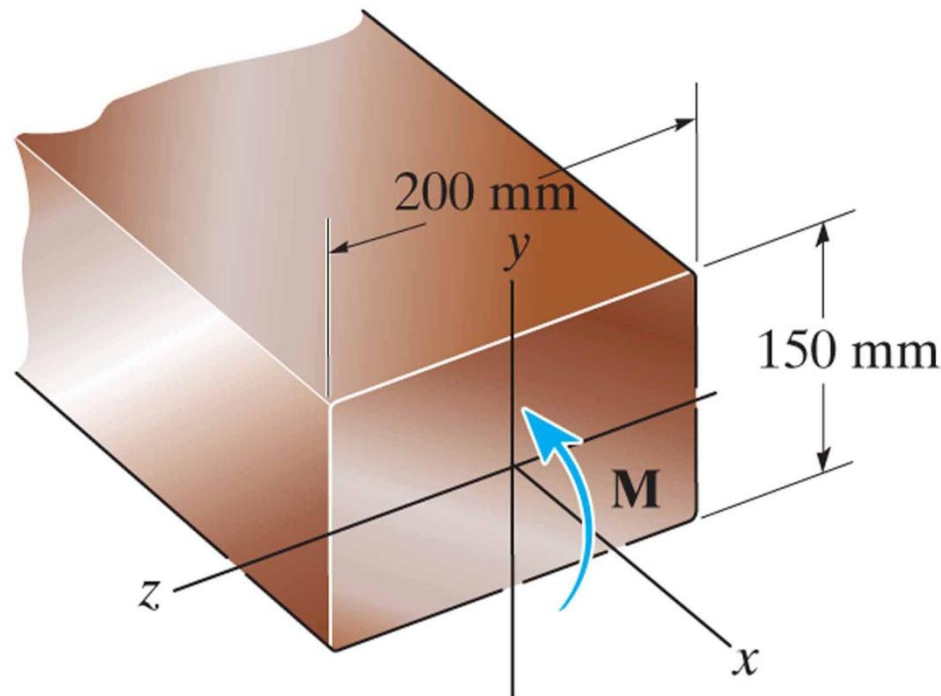
**Important to  
remember!!**





## Shear and bending diagrams: example C

A member having the dimensions shown is used to resist an internal bending moment of  $M = 90 \text{ kN}\cdot\text{m}$ . Determine the maximum stress in the member if the moment is applied (a) about the  $z$ -axis (as shown); and (b) about the  $y$ -axis. Sketch the stress distribution for each case.



# Shear and bending diagrams: example C

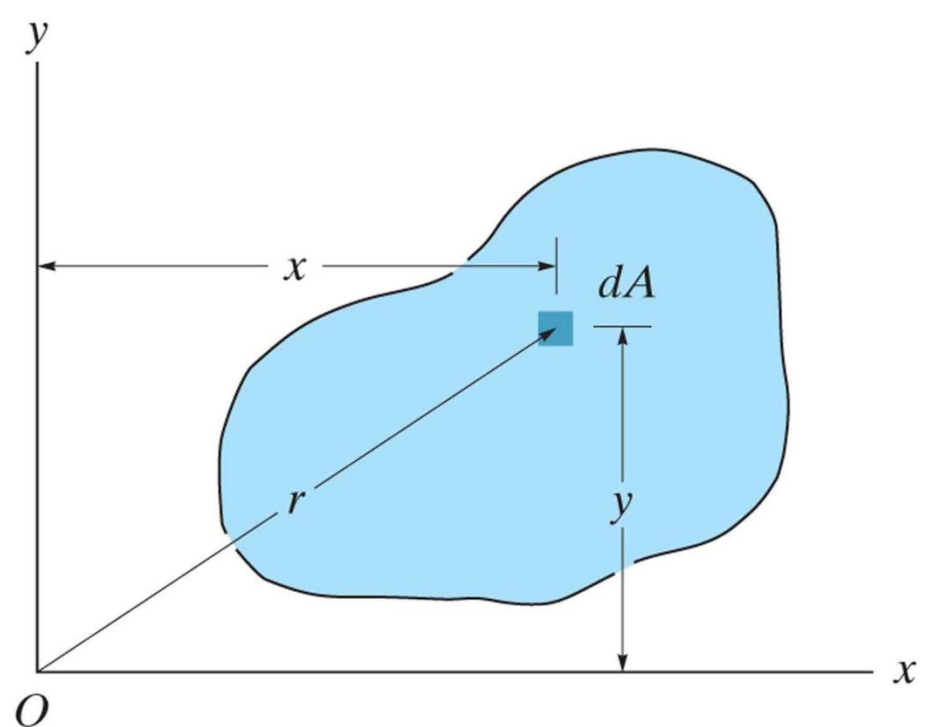
Area moment of inertia  
(**aka, 2<sup>nd</sup> area moment of inertia**):

$$I_{xx} = \int_A y^2 dA,$$

$$I_{yy} = \int_A x^2 dA$$

Polar area moment of inertia:

$$J_O = I_{xx} + I_{yy}$$



# Shear and bending diagrams: example C

Area moment of inertia:

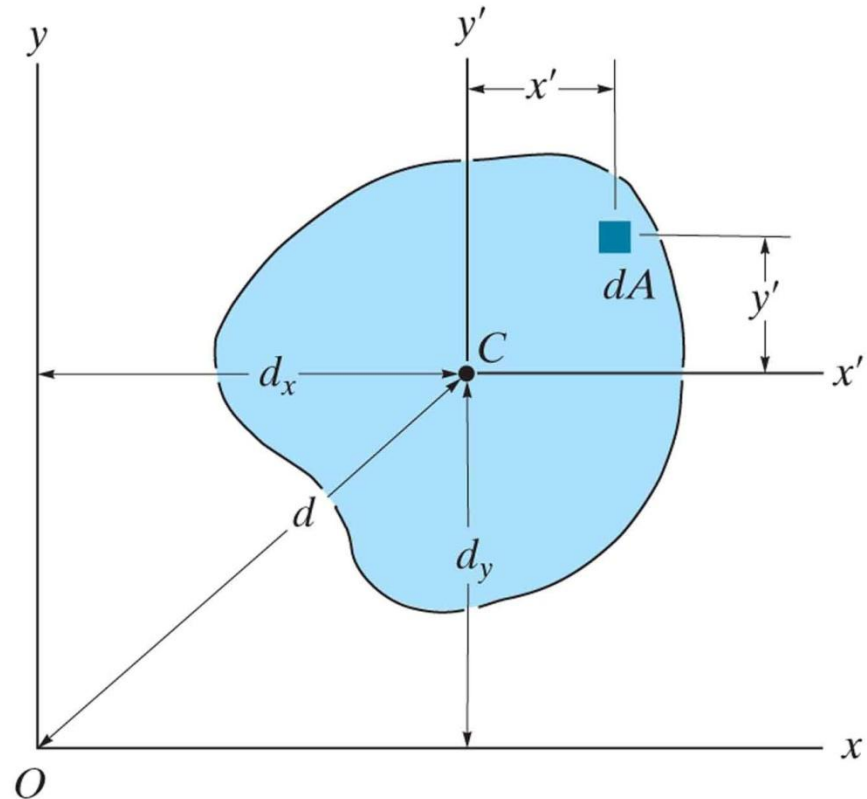
*parallel-axis theorem:*

$$I_{xx} = \bar{I}_{x'x'} + A \cdot d_y^2,$$

$$I_{yy} = \bar{I}_{y'y'} + A \cdot d_x^2$$

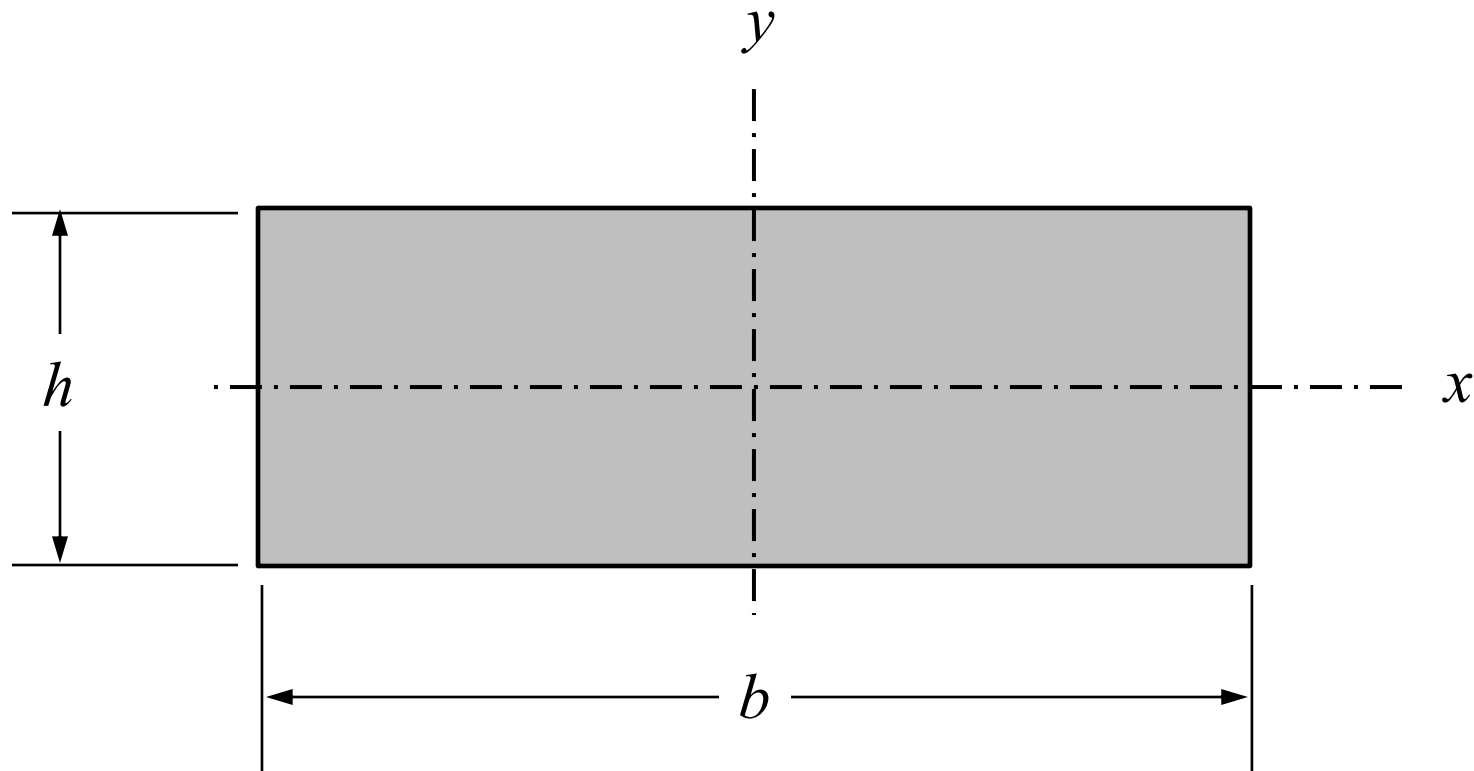
Polar area moment of inertia: *parallel-axis theorem:*

$$J_O = \bar{J}_C + A \cdot d^2$$



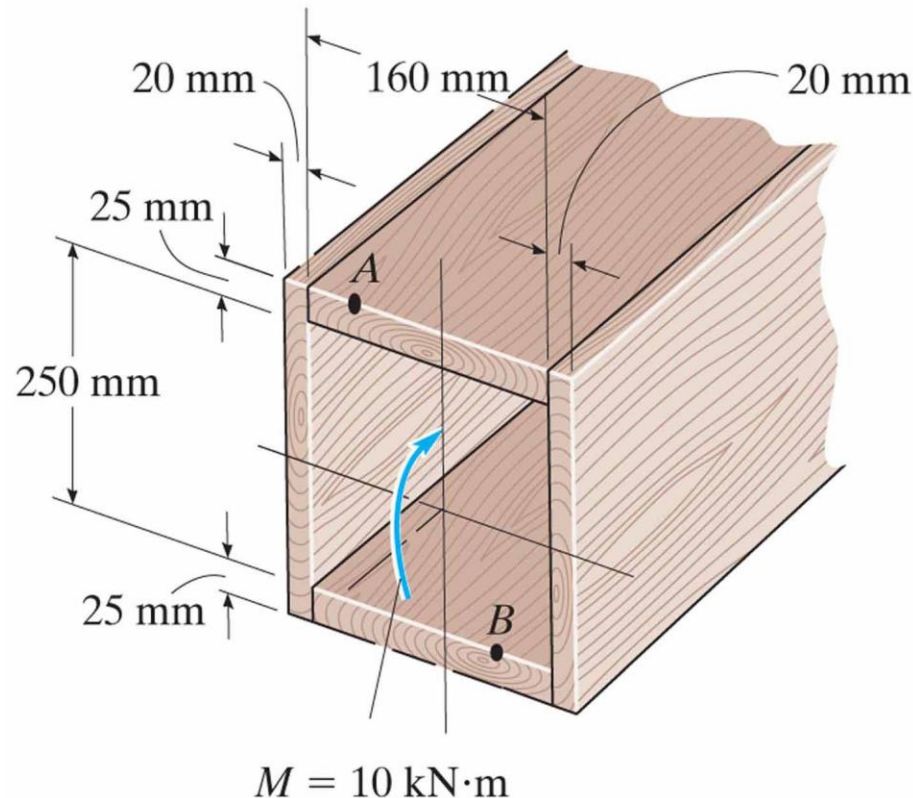
# Shear and bending diagrams: example C

$$I_{xx} = \frac{1}{12} b \cdot h^3, \quad I_{yy} = \frac{1}{12} h \cdot b^3$$



## Shear and bending diagrams: example D

A box beam is constructed from four pieces of wood, glued together as shown. If the moment acting on the cross-section is  $10 \text{ kN}\cdot\text{m}$ , determine the stresses at points  $A$  and  $B$  and show the results acting on volume elements located at these points.

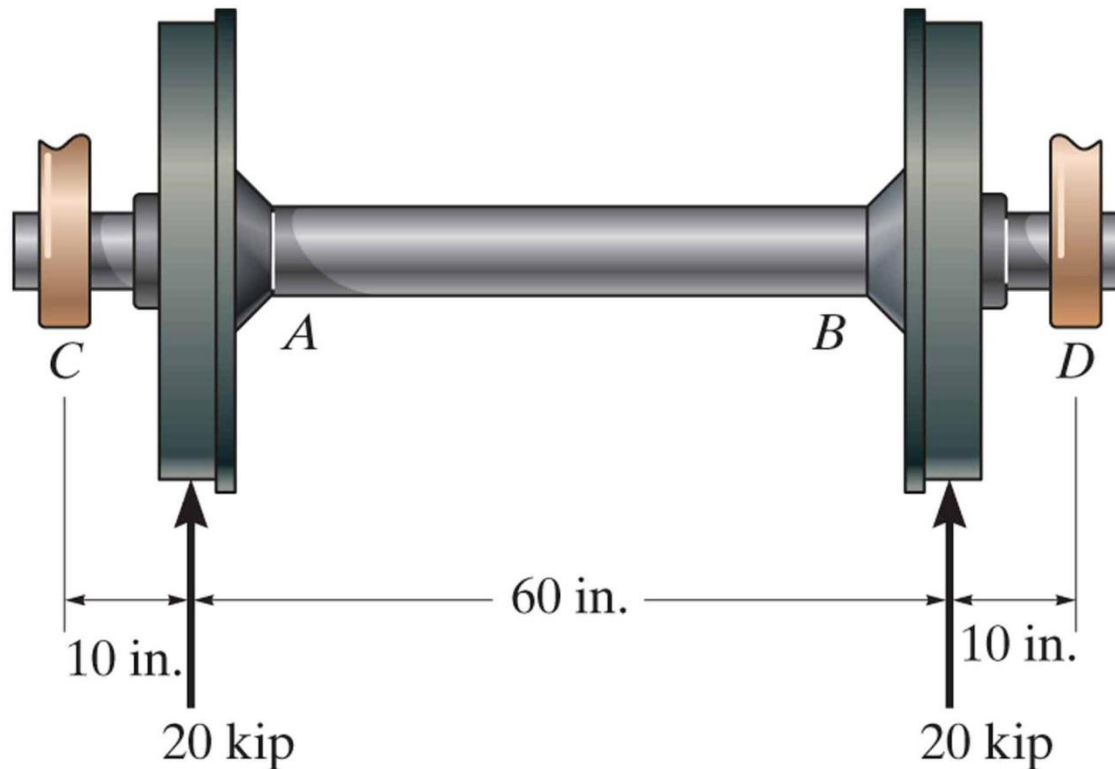


Note: state of stresses is *independent* of material (properties)



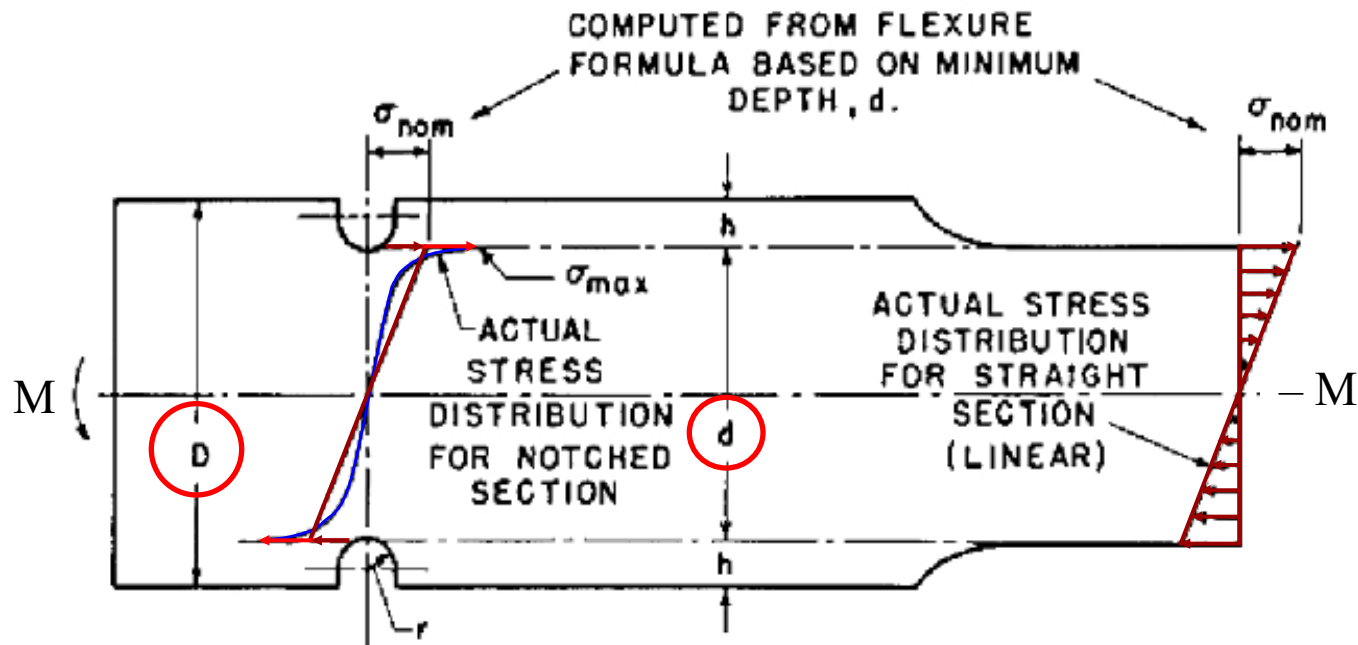
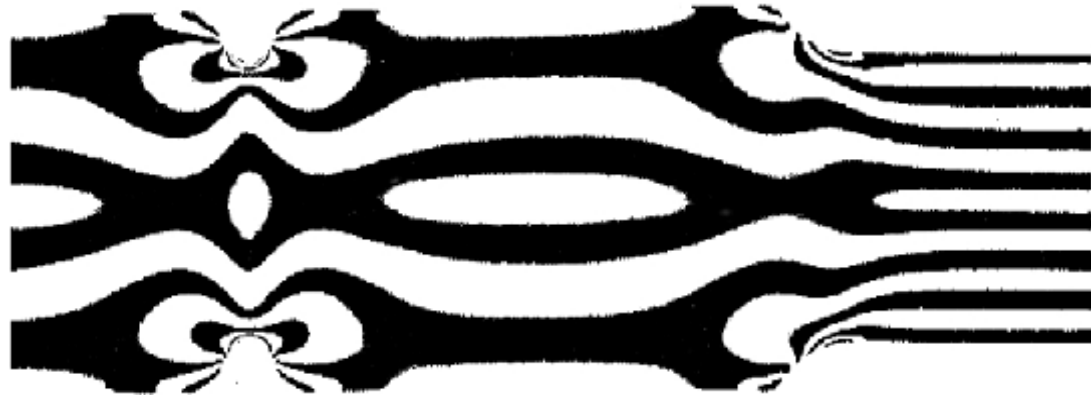
## Shear and bending diagrams: example E

The axle of the freight car is subjected to wheel loadings of 20 kip. If it is supported by two journal bearings at *C* and *D*, determine the maximum bending stress developed at the center of the axle, where the diameter is 5.5 in. **Apply two different methods**



# Stress concentrations: normal stresses: bending

Fringe pattern obtained with photoelasticity: pattern reveals distribution of internal stresses



# Stress concentrations: bending: normal stresses + shear

## Bending

Nominal  
bending  
stress:

$$\sigma_{nom} = \frac{Md}{I}$$

$$\sigma_{max} = K \frac{Md}{I}$$

$K$  is the stress  
concentration  
factor – normal  
stress

## Transversal shear

Nominal  
shear  
stress:

$$\tau_{nom}$$

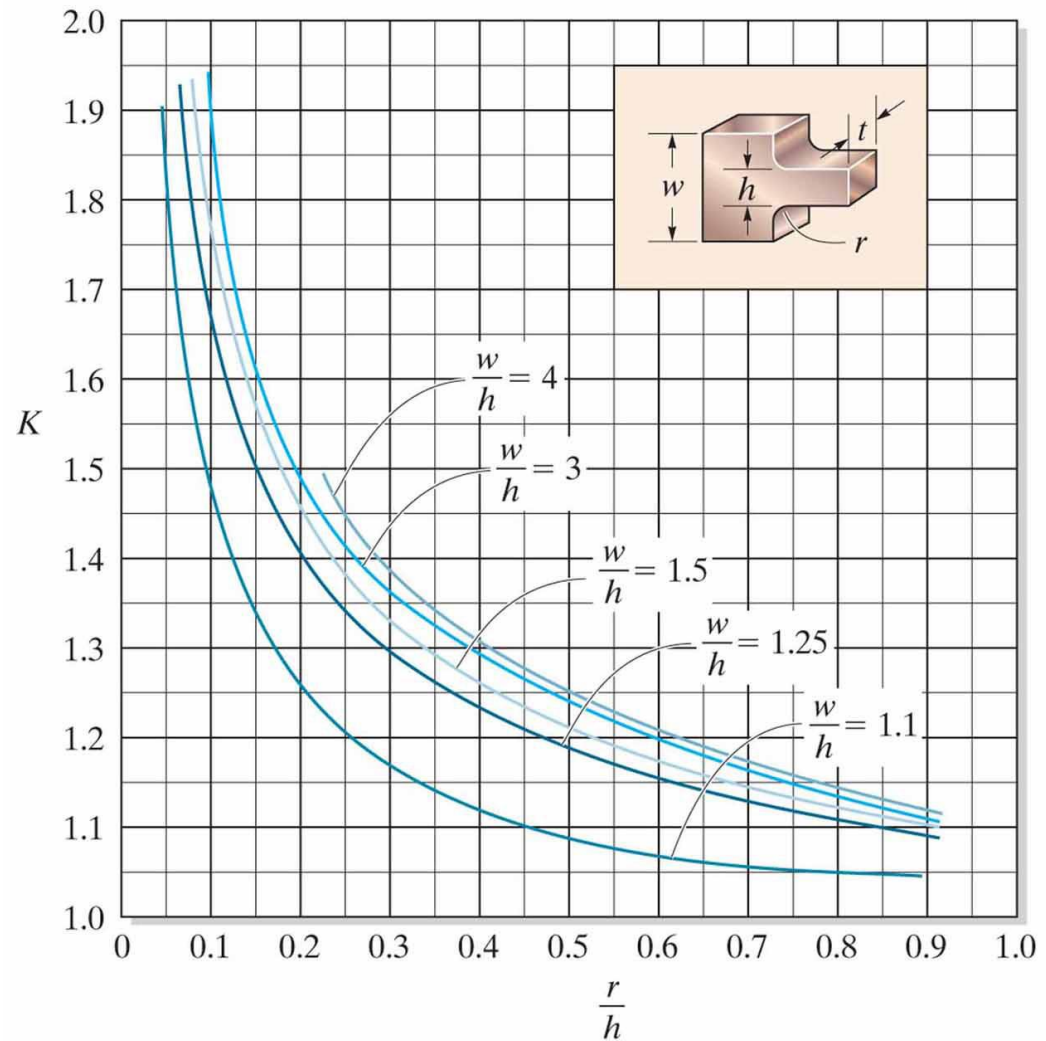
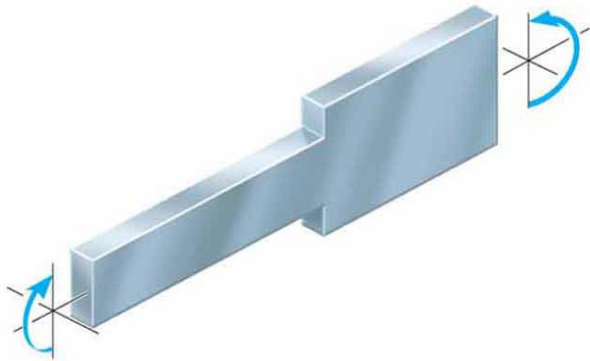
$$\tau_{max} = K_s \tau_{nom}$$

$K_s$  is the stress  
concentration factor  
– transversal shear  
stress

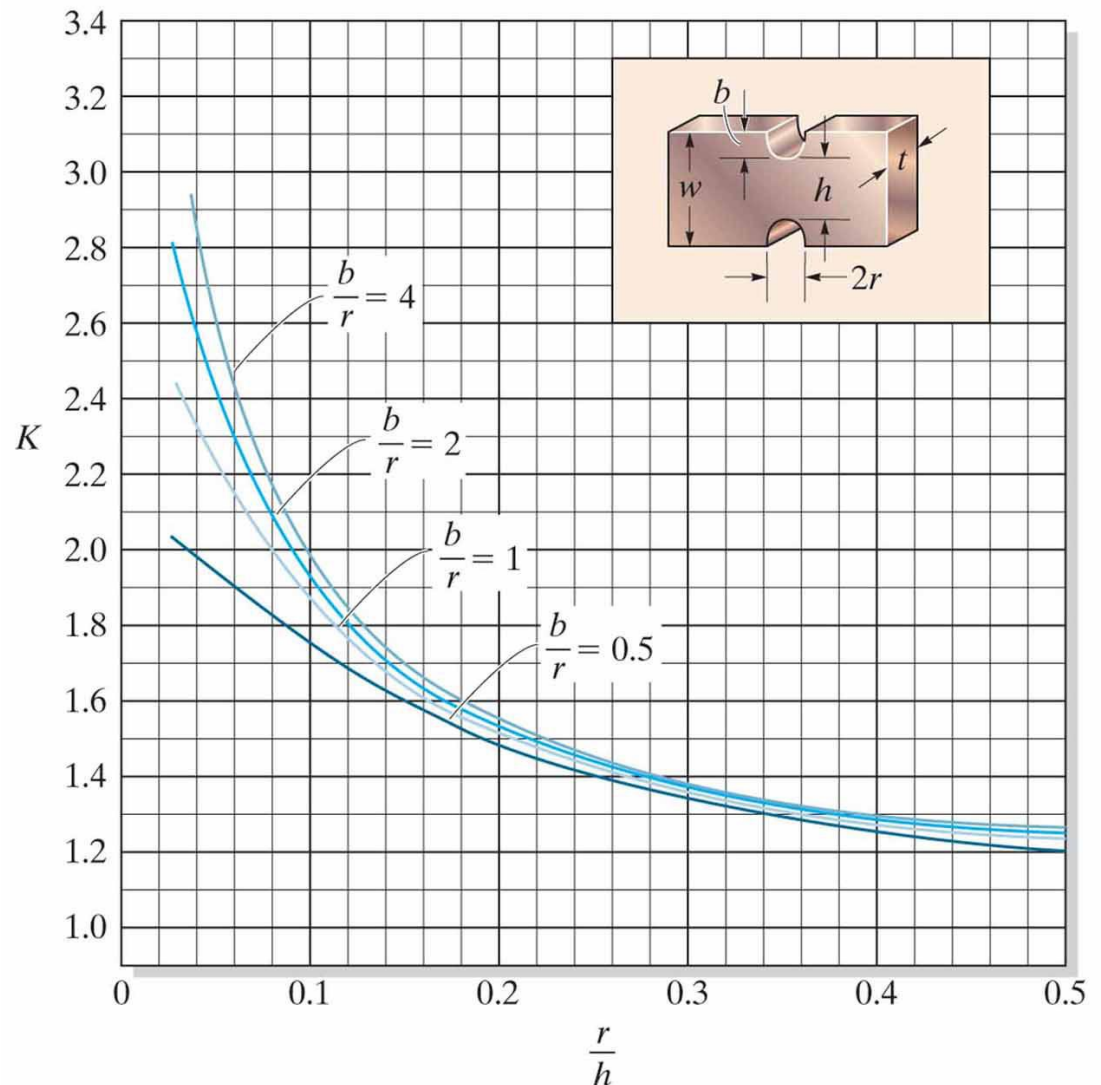
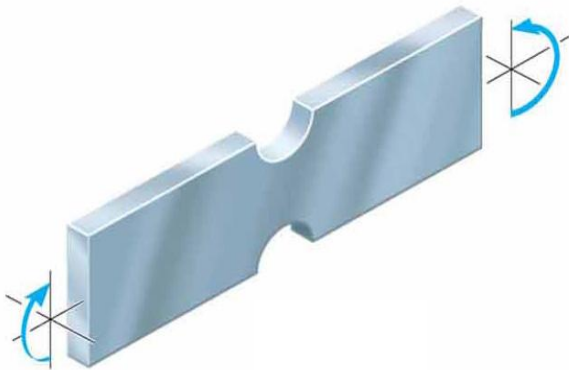




# Stress concentrations: normal stresses: bending



# Stress concentrations: normal stresses: bending



# Reading assignment

- Chapter 6 of textbook
- Review notes and text: ES2001, ES2501



# Homework assignment

- As indicated on webpage of our course

