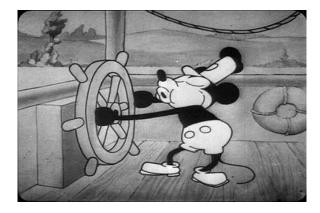
WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

STRESS ANALYSIS ES-2502, B'2025

We will get started soon...



23 October 2025





WORCESTER POLYTECHNIC INSTITUTE MECHANICAL ENGINEERING DEPARTMENT

STRESS ANALYSIS ES-2502, B'2025

Lecture 04:

Unit 3: definition of normal and shear stress

23 October 2025





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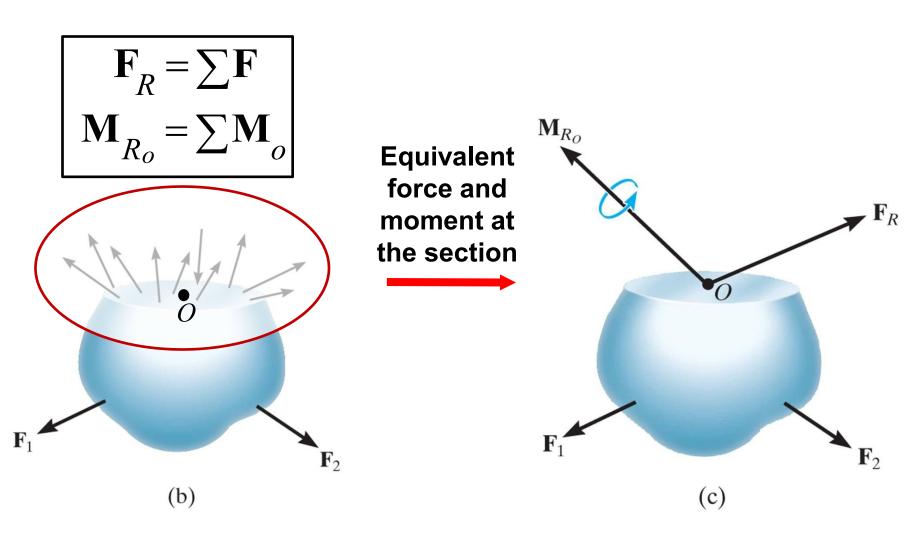
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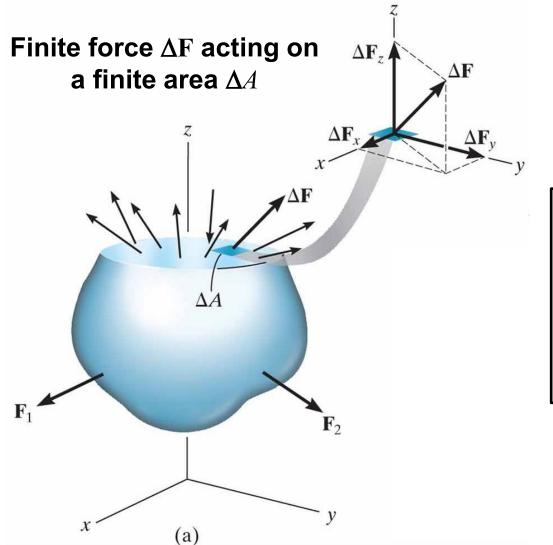












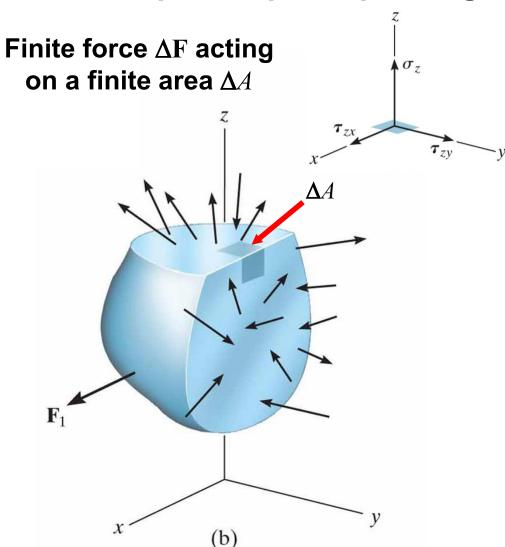
Definition

Normal stress:

$$\sigma_z = \lim_{\Delta A \to 0} \frac{\Delta F_z}{\Delta A}$$







Definitions

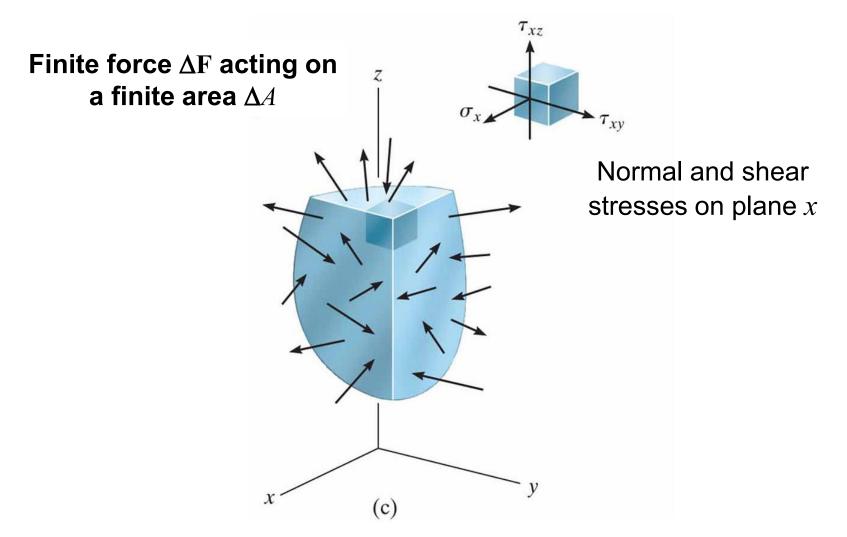
Shear stresses:

$$\tau_{zx} = \lim_{\Delta A \to 0} \frac{\Delta F_x}{\Delta A}$$

$$\tau_{zy} = \lim_{\Delta A \to 0} \frac{\Delta F_y}{\Delta A}$$



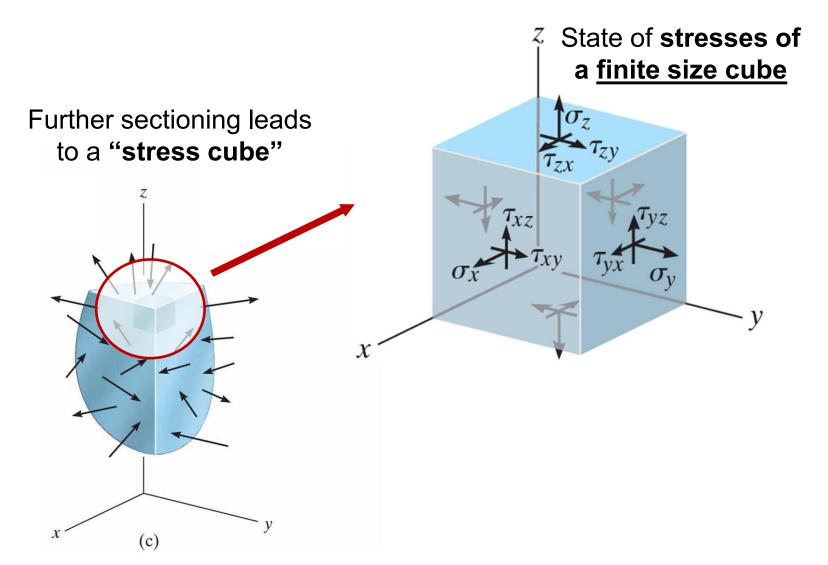








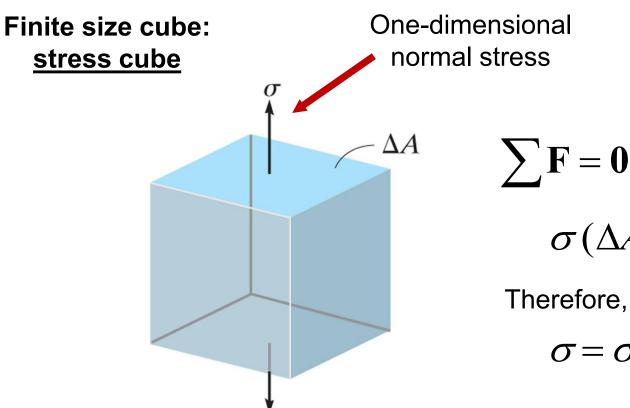
General state of stresses







State of stresses: normal stress equilibrium (1D)



$$\sum \mathbf{F} = \mathbf{0}; \implies$$

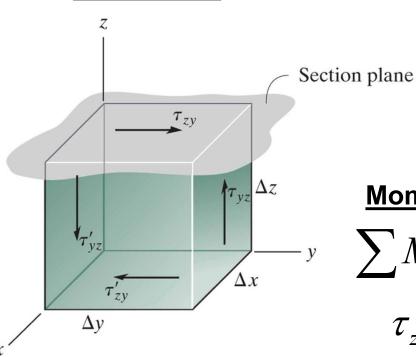
$$\sigma(\Delta A) = \sigma'(\Delta A)$$

$$\sigma = \sigma'$$



State of stresses: shear stress equilibrium

Finite size cube: stress cube



Forces:

$$\sum F_y = 0; \implies$$

$$\tau_{zy} \left(\Delta x \, \Delta y \right) = \tau'_{zy} \left(\Delta x \, \Delta y \right)$$

Therefore, $\tau_{zv} = \tau'_{zv}$

Moments:
$$\sum M_x = 0; \implies$$

$$\tau_{zy} \left(\Delta x \, \Delta y \right) \cdot \Delta z = \tau_{yz} \left(\Delta x \, \Delta z \right) \cdot \Delta y$$

Therefore, $au_{zv} = au_{vz}$





Tensile test

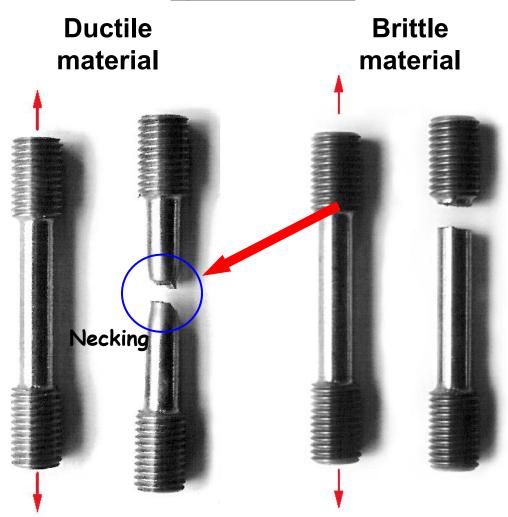




Tensile test

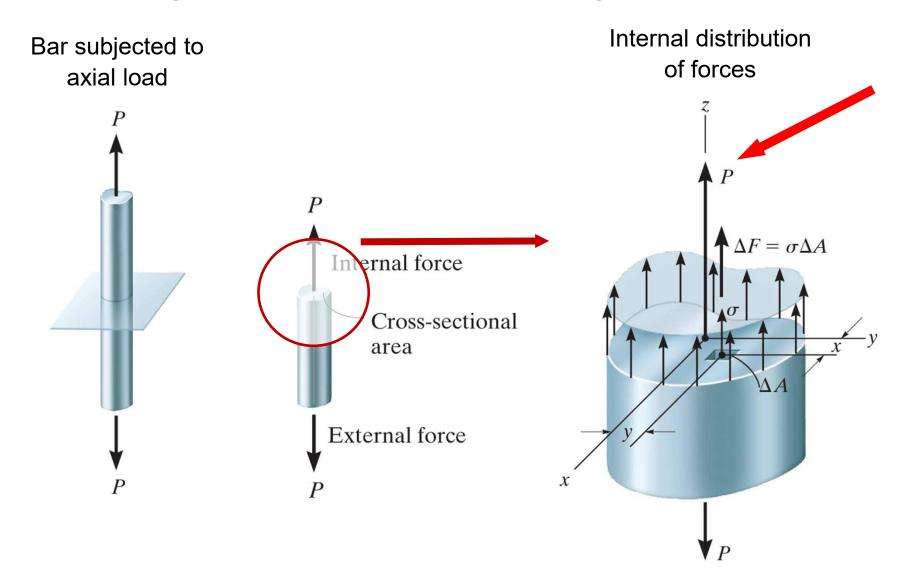


Typical results





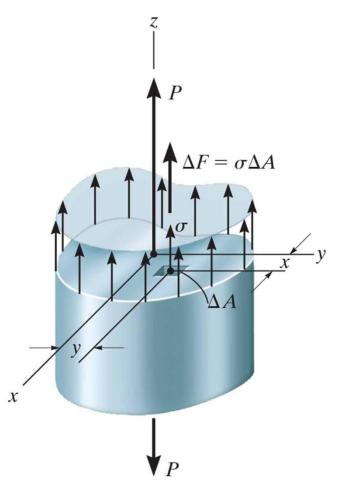








Internal distribution of forces



$$+ \uparrow F_{Rz} = \sum F_z$$

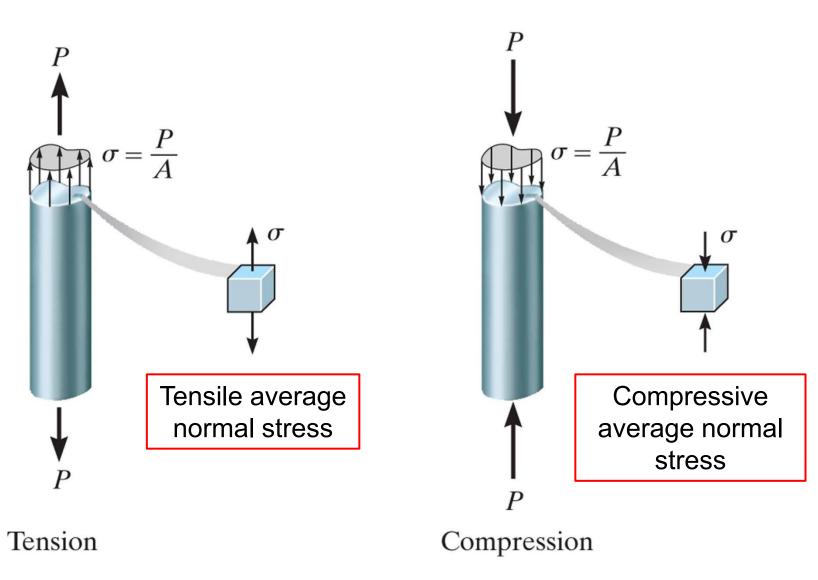
$$\int dF = \int_{A} \sigma \, dA$$
$$P = \sigma \, A$$

Average normal stress:

$$\sigma = \frac{P}{A}$$











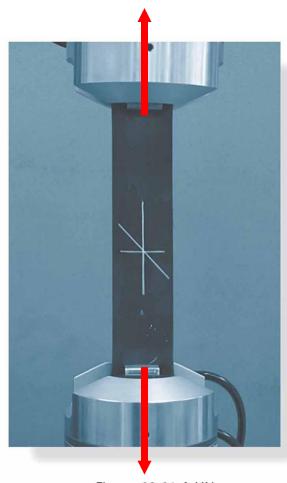


Figure: 02-01-A-UN

Note the before and after positions of three different line segments on this rubber membrane which is subjected to tension. The vertical line is lengthened, the horizontal line is shortened, and the inclined line changes its length and rotates. is shortened, and the inclined line changes its length and rotates.



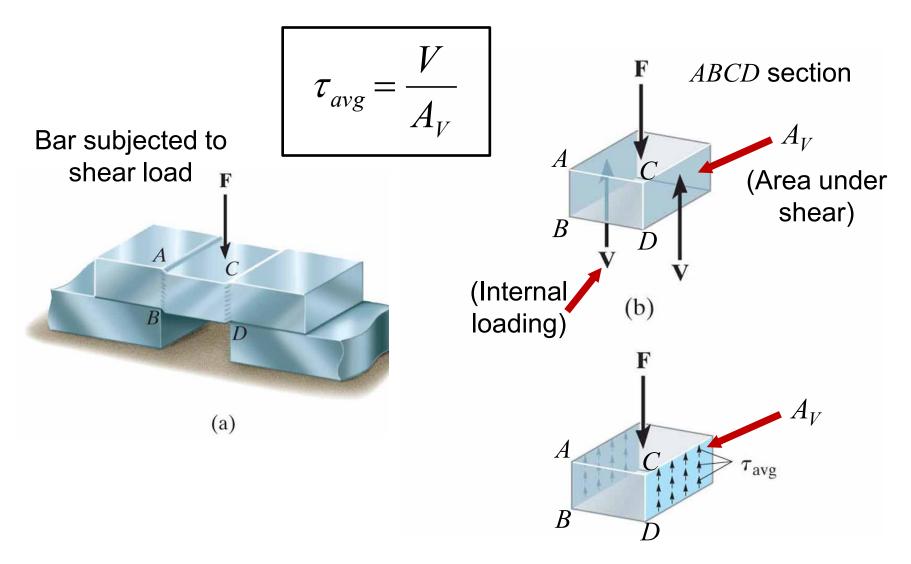
Figure: 02-01-B-UN

Note the before and after positions of three different line segments on this rubber membrane which is subjected to tension. The vertical line is lengthened, the horizontal line





Average direct shear stress

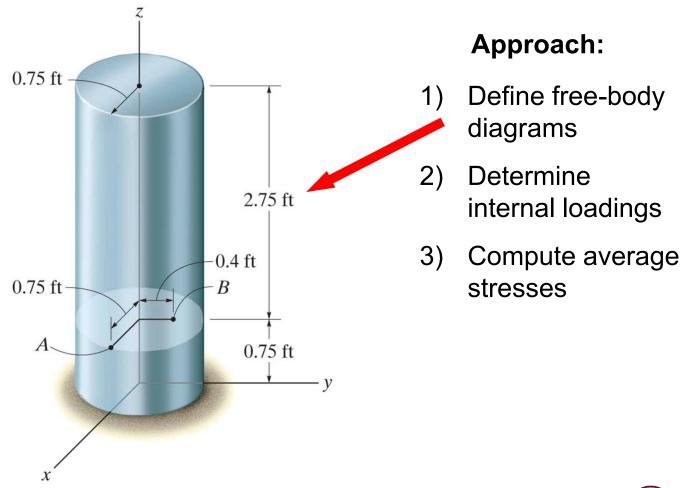






Average normal stress: example A

The casting shown is made of steel having a specific weight of γ_{st} =490 lb_f/ft^3 . Determine the average compressive stress acting at points A and B.

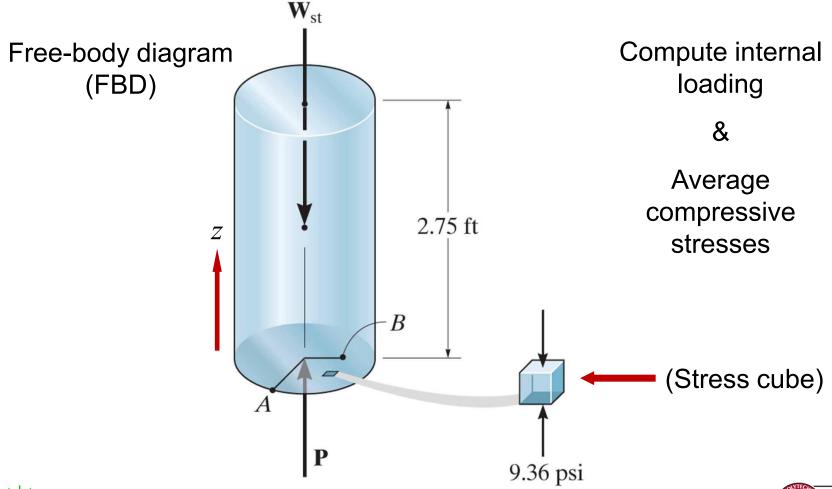






Average normal stress: example A

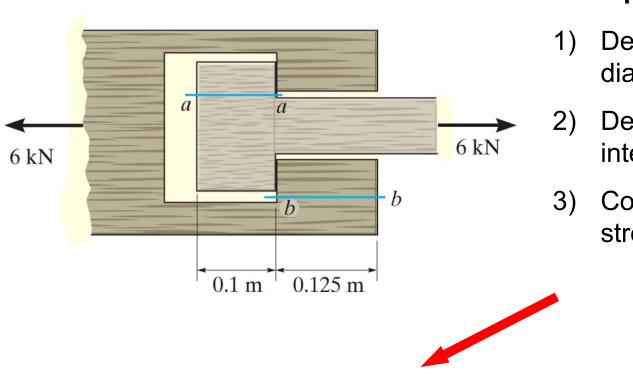
The casting shown is made of steel having a specific weight of $\gamma_{st}=490 \ lb_f/ft^3$. Determine the average compressive stress acting at points A and B.





Average shear stress: example B

Wood joints 150 mm deep (perpendicular to the plane) are loaded as shown. Determine the average shear stress developed along planes a-a and b-b.



Approach:

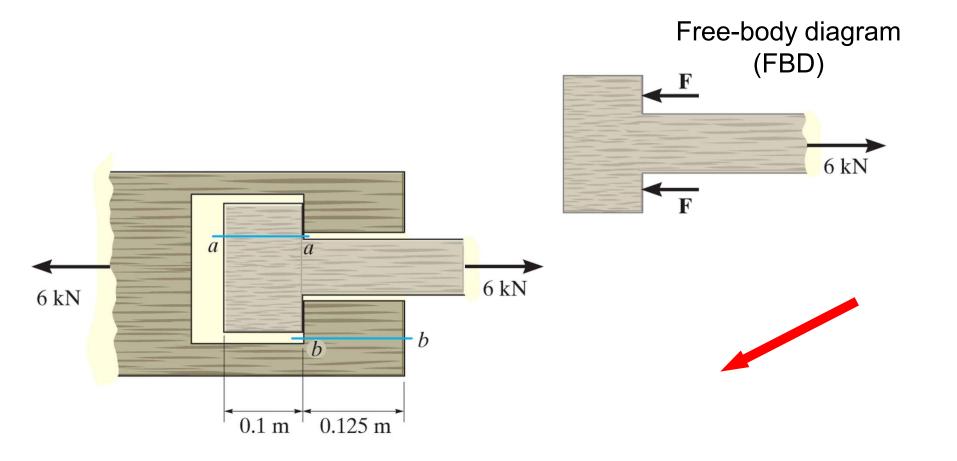
- Define free-body diagrams
- Determine internal loadings
- Compute average stresses





Average shear stress: example B

Wood joints 150 mm deep (perpendicular to the plane) are loaded as shown. Determine the average shear stress developed along planes a-a and b-b.

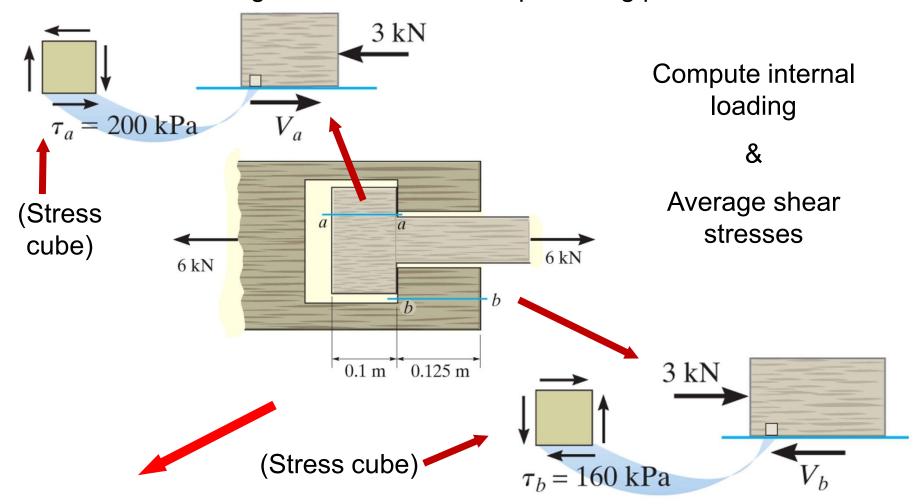






Average shear stress: example B

Wood joints 150 mm deep (perpendicular to the plane) are loaded as shown. Determine the average shear stress developed along planes a-a and b-b.





Reading assignment

- Chapter 1 of textbook
- Review notes and text: ES2001, ES2501





Homework assignment

As indicated on webpage of our course



