

Columns: $\frac{d^2y}{dx^2} + \frac{P}{EI}y = 0$ / Concentric

ME3320
Furlong

Recall: $y = C_1 \sin \sqrt{\frac{P}{EI}} x + C_2 \cos \sqrt{\frac{P}{EI}} x$

w/ C_1 and C_2 depend on BC's

for $y(x=0) = y(x=l) = 0 \Rightarrow C_2 = 0$

$$C_1 \sin \sqrt{\frac{P}{EI}} l = 0$$

$$\Rightarrow \sqrt{\frac{P}{EI}} l = n\pi$$

lowest load @ $n=1 \therefore \sqrt{\frac{P}{EI}} l = \pi$

$$P_{\text{critical}} = P = \frac{EI \pi^2}{l^2} \quad ; \text{ recall } I = A k^2$$

$$P_{\text{crit}} = \left(\frac{\pi}{l}\right)^2 \cdot E A k^2 = \frac{\pi^2 E A}{\left(\frac{l}{k}\right)^2} = \frac{\pi^2 E A}{S_r^2}$$

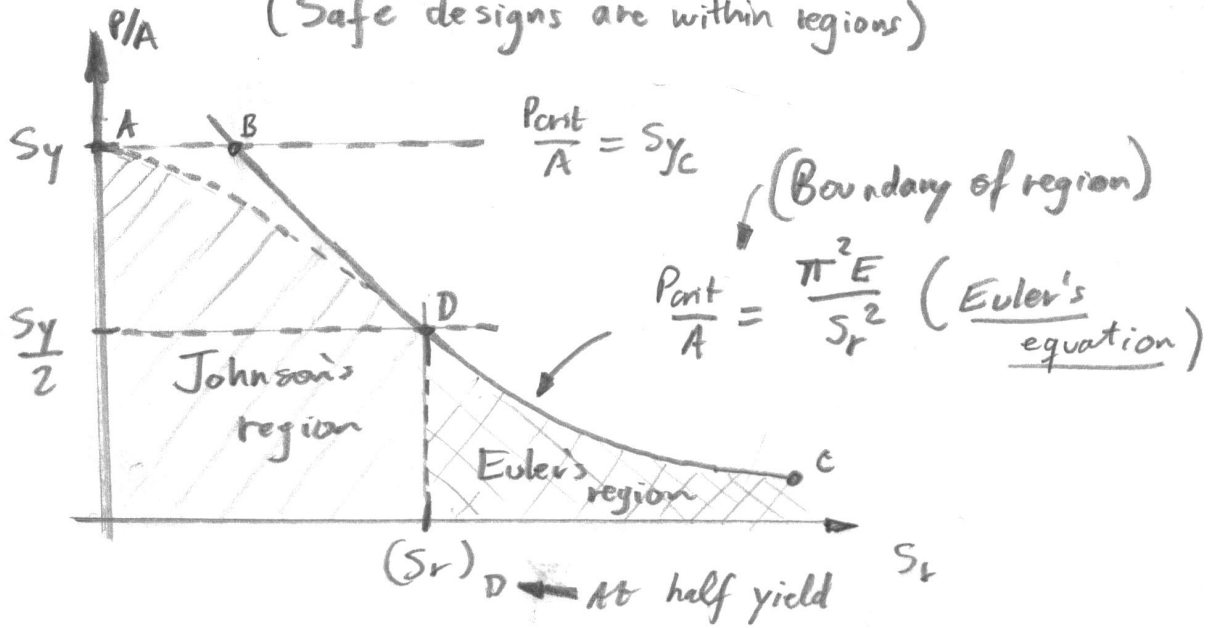
$$\frac{P_{\text{crit}}}{A} = \frac{\pi^2 E}{S_r^2}$$

which depends on BC's. To compensate for \neq BC's:

$$S_r = \frac{l_{\text{eff}}}{k} \quad ; \quad l_{\text{eff}} \text{ is in Table 4-4, pg 193}$$

Johnson & Euler regions

(Safe designs are within regions)



(1) Need to identify point D - to use Johnson's or Euler's formulations

(2) Set :
$$\frac{S_y}{2} = \frac{\pi^2 E}{S_r^2} \Rightarrow \text{Solve for } S_r \equiv (S_r)_D$$

$$(S_r)_D = \pi \sqrt{\frac{2E}{S_y}}$$

(3) Curve fit a parabola between points A and D
Use points A : $(0, S_y)$; B : $((S_r)_D, S_y/2)$

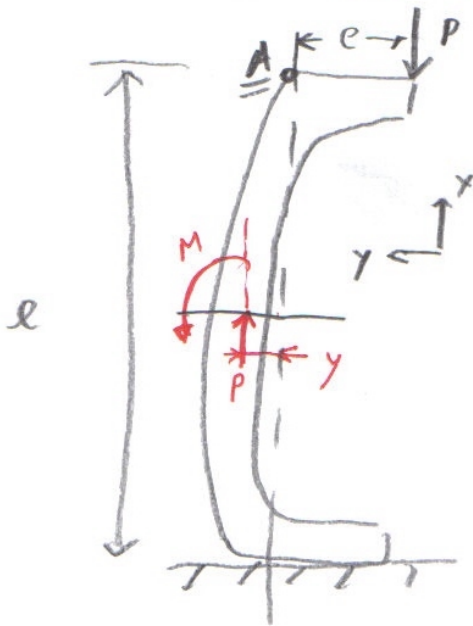
Curve fit leads to:

$$\frac{P_{crit}}{A} = S_y - \frac{1}{E} \left(\frac{S_y - S_r}{2\pi} \right)^2 ; \text{Johnson's equation}$$

Once P_{crit} is determined, calculate max. allowed forces

$$P_{allowed} = \frac{P_{critical}}{\text{Safety Factor}} = \frac{P_{crit}}{SF}$$

$SF > 1$; Need engg. standards (AISC, ASME, SAE, etc.)



$$\sum M_A = 0 = -M + P(e + y) = 0$$

Recall

$$M = EI \frac{d^2 y}{dx^2}$$



ODE

$$\frac{d^2 y}{dx^2} + \frac{P}{EI} y = -\frac{Pe}{EI}$$

BC's are:

$$y(x=0) = 0 \quad ; \quad \frac{dy}{dx}(x = \frac{l}{2}) = 0$$

$$\Rightarrow y(x = \frac{l}{2}) = e \left[\sec\left(\frac{l}{2} \sqrt{\frac{P}{EI}}\right) - 1 \right]$$

with max. moment

$$M_{\max} = -P(e + y_{l/2}) = -P \cdot e \cdot \sec\left(\frac{l}{2} \sqrt{\frac{P}{EI}}\right)$$

Comp. stress is

$$\sigma_c = \frac{P}{A} - \frac{Mc}{I} = \frac{P}{A} - \frac{Mc}{AK^2}$$

$$\text{Using } M = \text{Max} \Rightarrow \sigma_c = \frac{P}{A} \left[1 + \left(\frac{e \cdot c}{K^2}\right) \sec\left(\frac{l}{K} \sqrt{\frac{P}{4EA}}\right) \right]$$

Using $\sigma_c = S_{yc}$ (Yield strength in compression)

$$\frac{P}{A} = \frac{S_{yc}}{1 + \left(\frac{e \cdot c}{K^2}\right) \sec\left(\frac{l_{\text{eff}}}{K} \sqrt{\frac{P}{4EA}}\right)}$$

(Note that P is in both sides of Eq.)

PROBLEM 4-51a

Statement: A solid, rectangular column has the dimensions and properties below. Determine if it is a Johnson or an Euler column and find the critical load:

- (a) If its boundary conditions are pinned-pinned.
 (b) If its boundary conditions are fixed-pinned.
 (c) If its boundary conditions are fixed-fixed.
 (d) If its boundary conditions are fixed-free.

Units: $N := \text{newton}$ $MPa := 10^6 \cdot Pa$ $GPa := 10^9 \cdot Pa$ $kN := 10^3 \cdot N$

Given: Length of column $L := 100 \text{ mm}$ Material Steel
 Thickness $t := 10 \text{ mm}$ Yield strength $S_y := 300 \text{ MPa}$
 Height $h := 20 \text{ mm}$ Modulus of elasticity $E := 207 \text{ GPa}$

Solution: See Mathcad file P0451a.

1. Calculate the slenderness ratio that divides the unit load vs slenderness ratio graph into Johnson and Euler regions.

$$S_{yD} := \pi \sqrt{\frac{2 \cdot E}{S_y}} \qquad S_{yD} = 116.7$$

2. Calculate the cross-section area, moment of inertia, and the radius of gyration.

Area $A := h \cdot t$ $A = 200.00 \text{ mm}^2$

Moment of inertia $I := \frac{h \cdot t^3}{12}$ $I = 1667 \text{ mm}^4$

Radius of gyration $k := \sqrt{\frac{I}{A}}$ $k = 2.887 \text{ mm}$

3. Define functions to determine column type and critical load.

Type $\text{type}(S_y) := \begin{cases} \text{"Euler"} & \text{if } S_y > S_{yD} \\ \text{"Johnson"} & \text{otherwise} \end{cases}$

Critical load $P_{cr}(S_y) := \begin{cases} \text{return } A \cdot \frac{\pi^2 \cdot E}{S_y^2} & \text{if } \text{type}(S_y) = \text{"Euler"} \\ A \cdot \left[S_y - \frac{1}{E} \cdot \left(\frac{S_y \cdot S_y}{2 \cdot \pi} \right)^2 \right] & \text{otherwise} \end{cases}$

(a) pinned-pinned ends

4. Using Table 4-7, calculate the effective column length.

$L_{eff} := 1 \cdot L$ $L_{eff} = 100 \text{ mm}$

5. Calculate the slenderness ratio for the column.

Slenderness ratio $S_y := \frac{L_{eff}}{k}$ $S_y = 34.64$

6. Determine the type and critical load using the functions defined above.

$$\text{type}(S_y) = \text{"Johnson"} \quad P_{cr}(S_y) = 57.36 \text{ kN}$$

(b) fixed-pinned ends

7. Using Table 4-7, calculate the effective column length.

$$L_{eff} := 0.8 \cdot L \quad L_{eff} = 80 \text{ mm}$$

8. Calculate the slenderness ratio for the column.

$$\text{Slenderness ratio} \quad S_y := \frac{L_{eff}}{k} \quad S_y = 27.71$$

9. Determine the type and critical load using the functions defined above.

$$\text{type}(S_y) = \text{"Johnson"} \quad P_{cr}(S_y) = 58.31 \text{ kN}$$

(c) fixed-fixed ends

10. Using Table 4-7, calculate the effective column length.

$$L_{eff} := 0.65 \cdot L \quad L_{eff} = 65 \text{ mm}$$

11. Calculate the slenderness ratio for the column.

$$\text{Slenderness ratio} \quad S_y := \frac{L_{eff}}{k} \quad S_y = 22.52$$

12. Determine the type and critical load using the functions defined above.

$$\text{type}(S_y) = \text{"Johnson"} \quad P_{cr}(S_y) = 58.9 \text{ kN}$$

(d) fixed-free ends

13. Using Table 4-7, calculate the effective column length.

$$L_{eff} := 2.1 \cdot L \quad L_{eff} = 210 \text{ mm}$$

13. Calculate the slenderness ratio for the column.

$$\text{Slenderness ratio} \quad S_y := \frac{L_{eff}}{k} \quad S_y = 72.75$$

14. Determine the type and critical load using the functions defined above.

$$\text{type}(S_y) = \text{"Johnson"} \quad P_{cr}(S_y) = 48.34 \text{ kN}$$

PROBLEM 4-52a

Statement: A solid, circular column, loaded eccentrically, has the dimensions and properties below. Find the critical load:

- (a) If its boundary conditions are pinned-pinned.
 (b) If its boundary conditions are fixed-pinned.
 (c) If its boundary conditions are fixed-fixed.
 (d) If its boundary conditions are fixed-free.

Units: $N := \text{newton}$ $MPa := 10^6 \cdot Pa$ $GPa := 10^9 \cdot Pa$ $kN := 10^3 \cdot N$

Given: Length of column $L := 100 \text{ mm}$ Material Steel
 Outside diameter $od := 20 \text{ mm}$ Yield strength $S_y := 300 \text{ MPa}$
 Eccentricity (t) $e := 10 \text{ mm}$ Modulus of elasticity $E := 207 \text{ GPa}$

Solution: See Mathcad file P0452a.

1. Calculate the cross-section area, distance to extreme fiber, and the moment of inertia.

Area $A := \frac{\pi}{4} \cdot od^2$ $A = 314.16 \text{ mm}^2$
 Distance to extreme fiber $c := 0.5 \cdot od$ $c = 10 \text{ mm}$
 Moment of inertia $I := \frac{\pi}{64} \cdot od^4$ $I = 7854 \text{ mm}^4$

4. Calculate the radius of gyration and eccentricity ratio for the column.

Radius of gyration $k := \sqrt{\frac{I}{A}}$ $k = 5.00 \text{ mm}$
 Eccentricity ratio $E_r := \frac{e \cdot c}{k^2}$ $E_r = 4.0$

(a) pinned-pinned ends

3. Using Table 4-7, calculate the effective column length.

$L_{eff} := 1 \cdot L$ $L_{eff} = 100 \text{ mm}$

4. Calculate the slenderness ratio for the column.

Slenderness ratio $S_r := \frac{L_{eff}}{k}$ $S_r = 20.00$

5. Calculate the critical load using the Secant equation.

Guess $P := 1 \text{ kN}$

Given

$$P = \frac{S_y \cdot A}{1 + E_r \cdot \sec\left(S_r \cdot \sqrt{\frac{P}{4 \cdot E \cdot A}}\right)}$$

$P_{cr} := \text{Find}(P)$ $P_{cr} = 18.63 \text{ kN}$

(b) fixed-pinned ends

6. Using Table 4-7, calculate the effective column length.

$$L_{eff} := 0.8 \cdot L$$

$$L_{eff} = 80 \text{ mm}$$

7. Calculate the slenderness ratio for the column.

$$\text{Slenderness ratio} \quad S_y := \frac{L_{eff}}{k} \quad S_y = 16.00$$

8. Calculate the critical load using the Secant equation.

$$\text{Guess} \quad P := 1 \text{ kN}$$

Given

$$P = \frac{S_y \cdot A}{1 + E_y \cdot \sec\left(S_y \sqrt{\frac{P}{4 \cdot E \cdot A}}\right)}$$

$$P_{cr} := \text{Find}(P) \quad P_{cr} = 18.71 \text{ kN}$$

(c) fixed-fixed ends

9. Using Table 4-7, calculate the effective column length.

$$L_{eff} := 0.65 \cdot L$$

$$L_{eff} = 65 \text{ mm}$$

10. Calculate the slenderness ratio for the column.

$$\text{Slenderness ratio} \quad S_y := \frac{L_{eff}}{k} \quad S_y = 13.00$$

11. Calculate the critical load using the Secant equation.

$$\text{Guess} \quad P := 1 \text{ kN}$$

Given

$$P = \frac{S_y \cdot A}{1 + E_y \cdot \sec\left(S_y \sqrt{\frac{P}{4 \cdot E \cdot A}}\right)}$$

$$P_{cr} := \text{Find}(P) \quad P_{cr} = 18.76 \text{ kN}$$

(d) fixed-free ends

12. Using Table 4-7, calculate the effective column length.

$$L_{eff} := 2.1 \cdot L$$

$$L_{eff} = 210 \text{ mm}$$

13. Calculate the slenderness ratio for the column.

$$\text{Slenderness ratio} \quad S_y := \frac{L_{eff}}{k} \quad S_y = 42$$

14. Calculate the critical load using the Secant equation.

$$\text{Guess} \quad P := 1 \text{ kN}$$

Given

$$P = \frac{S_y \cdot A}{1 + E_y \cdot \sec\left(S_y \sqrt{\frac{P}{4 \cdot E \cdot A}}\right)}$$

$$P_{cr} := \text{Find}(P) \quad P_{cr} = 17.93 \text{ kN}$$